

Wobble / Nutating Plate Pumps

Project(s) review

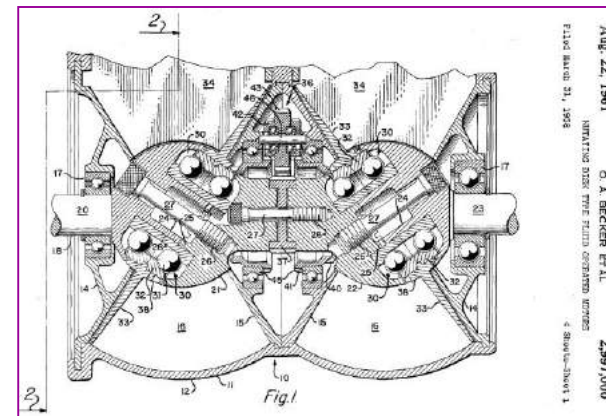
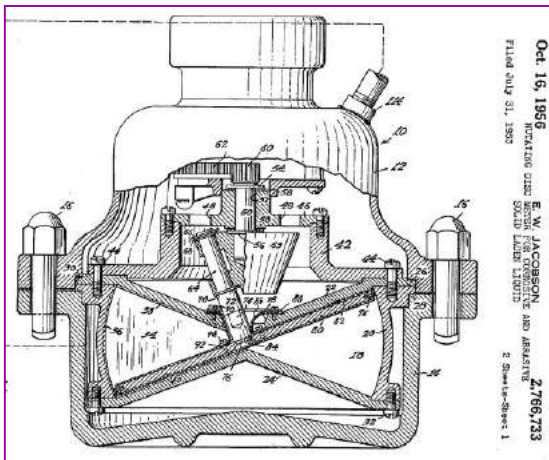
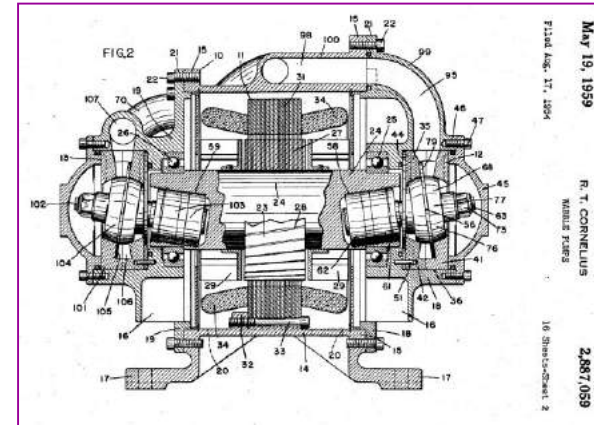
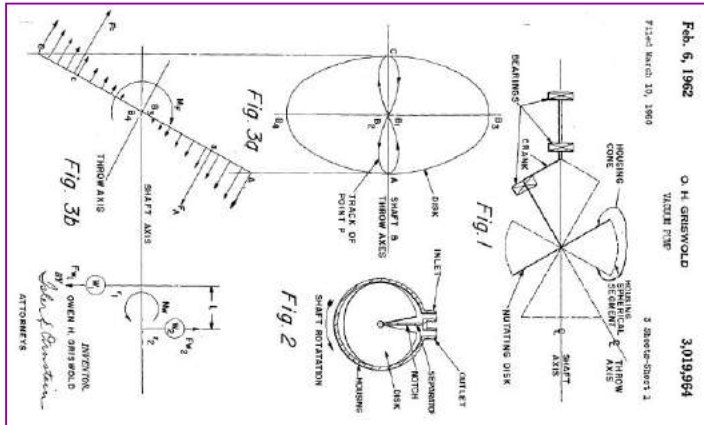
By Lindsay Dalziel, Design Director of

Design SMART® Ltd

Innovative Mechanical Engineering

www.DesignSMART.co.nz

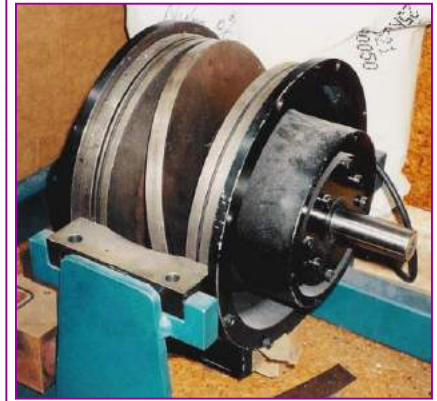
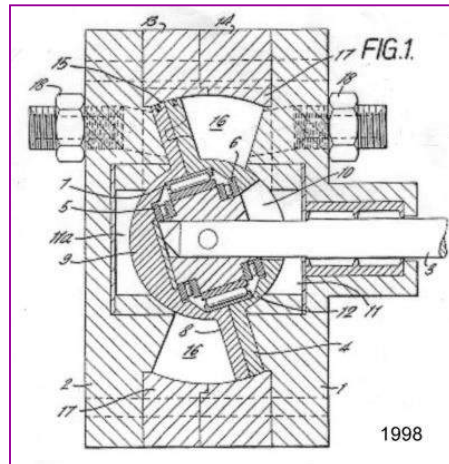
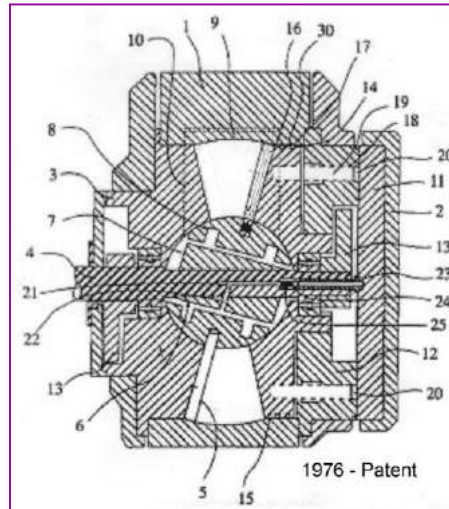
History of Wobble Plate Pumps



All these above designed without modern CAD/CAM & Simulation tools, however from what we know now this would have limited their progress, IP drawings shown above are from the 1950's & 1960's. They all utilizes the slant axis principle which has been around since 1896, (the oldest IP that we discovered).

Wobble Plate Pumps, the NZ History

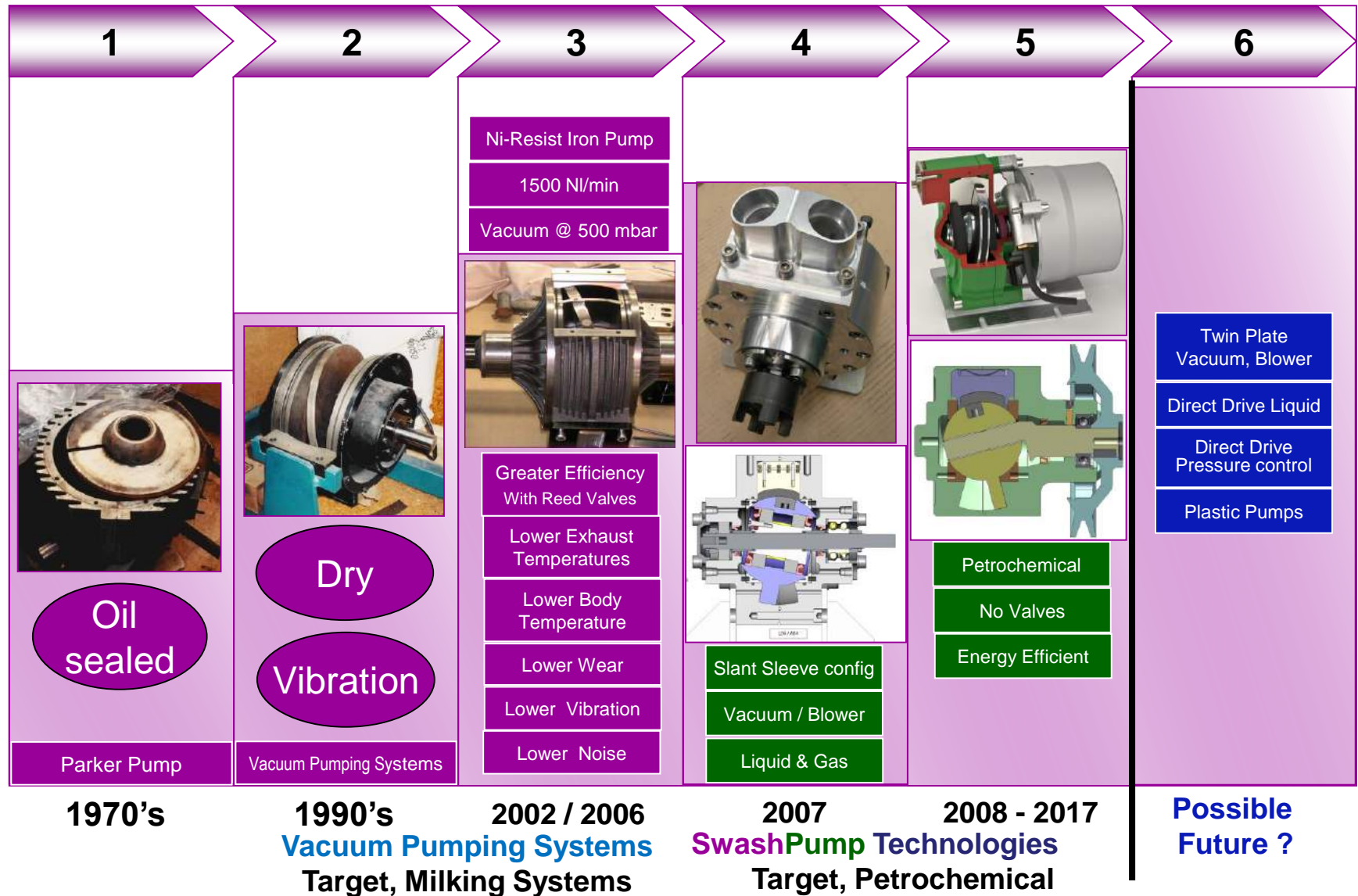
- Was firstly know in NZ as the Parker Pump, named after Alf Parker, an ex British World War II Royal Air force engineer who immigrated to NZ.
- This slant axis mechanism was adapted by Parker into a Vacuum Pump in New Zealand in the 1970's (without CAD CAM).
- He lodged his first Patent in 1976, his last in 1998 just prior to his death.
- Parker ran his pump(s) oil lubricated (as a sealing agent) rather than as a dry running vacuum pump.
- Vacuum Pumping Systems, partnered with him in 1995 & converted it into a non lubricated dry running Vacuum Pump & continued the development.
- SwashPump Technologies then took the IP & made significant inventive step to the IP between 2007 & 2017.



Resources used during recent NZ development

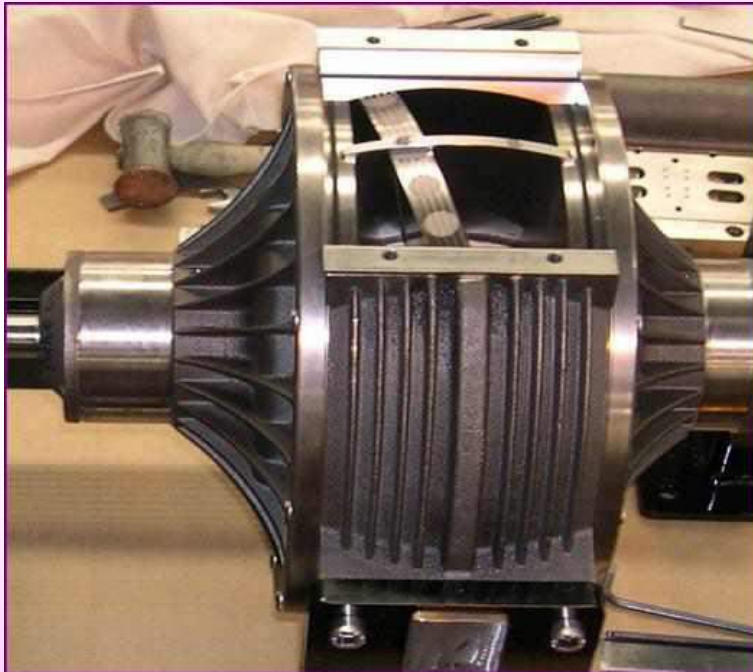
- ➡ SolidWorks 2002 through to SolidWorks 2016.
- ➡ SolidWorks Simulation (FEA) in all of the above.
- ➡ SolidWorks PDMWE, 3 full users & 3 contributors, (2007 to 2012).
- ➡ SolidWorks Motion / Mechanism design, re dynamic balance.
Also Industrial Research support in the early development.
- ➡ CFD support from Paul @ Matrix Applied Computing in 2008,
using STAR-CCM+ from CD-adapco.
- ➡ NC manufacturing & tool making by RPM and Axiam Engineering.
- ➡ Injection (over) moulding by Galantai & Absolute Plastics.
- ➡ CMM QA support from Brendan @ Axiam Engineering,
who are also SolidWorks users.

Wobble Plate Pumps (NZ) History continued



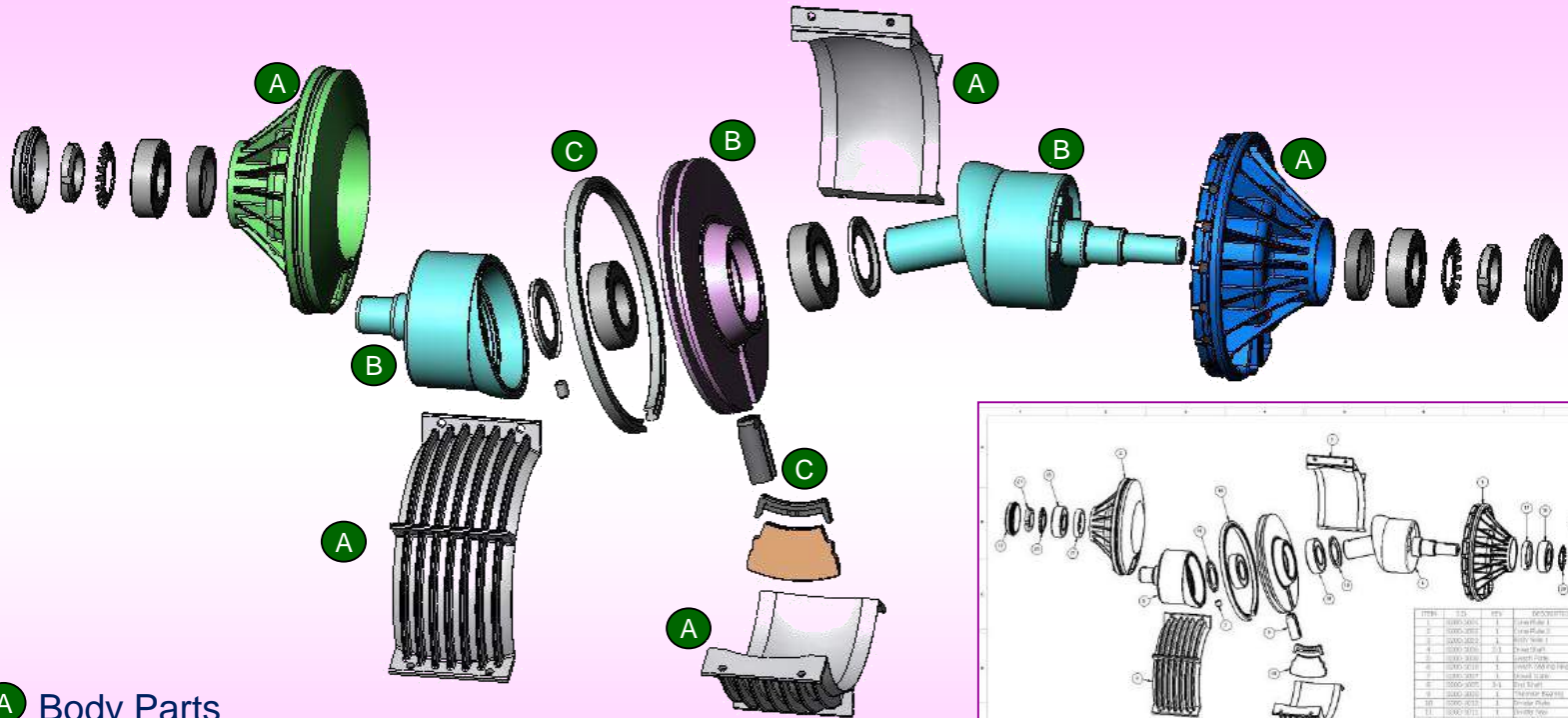
Vacuum Pumping Systems, Version 03 Pump

500 mbar 1500 NI/min Vacuum Pump
Target market was milking systems (2004)

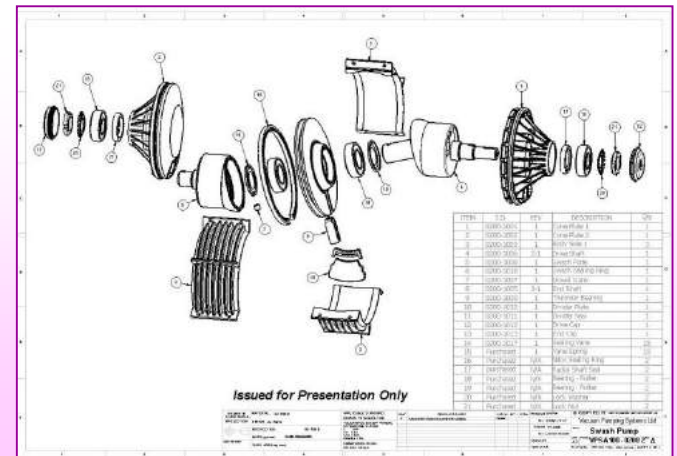


The Version 03 (Vacuum) Pump Design

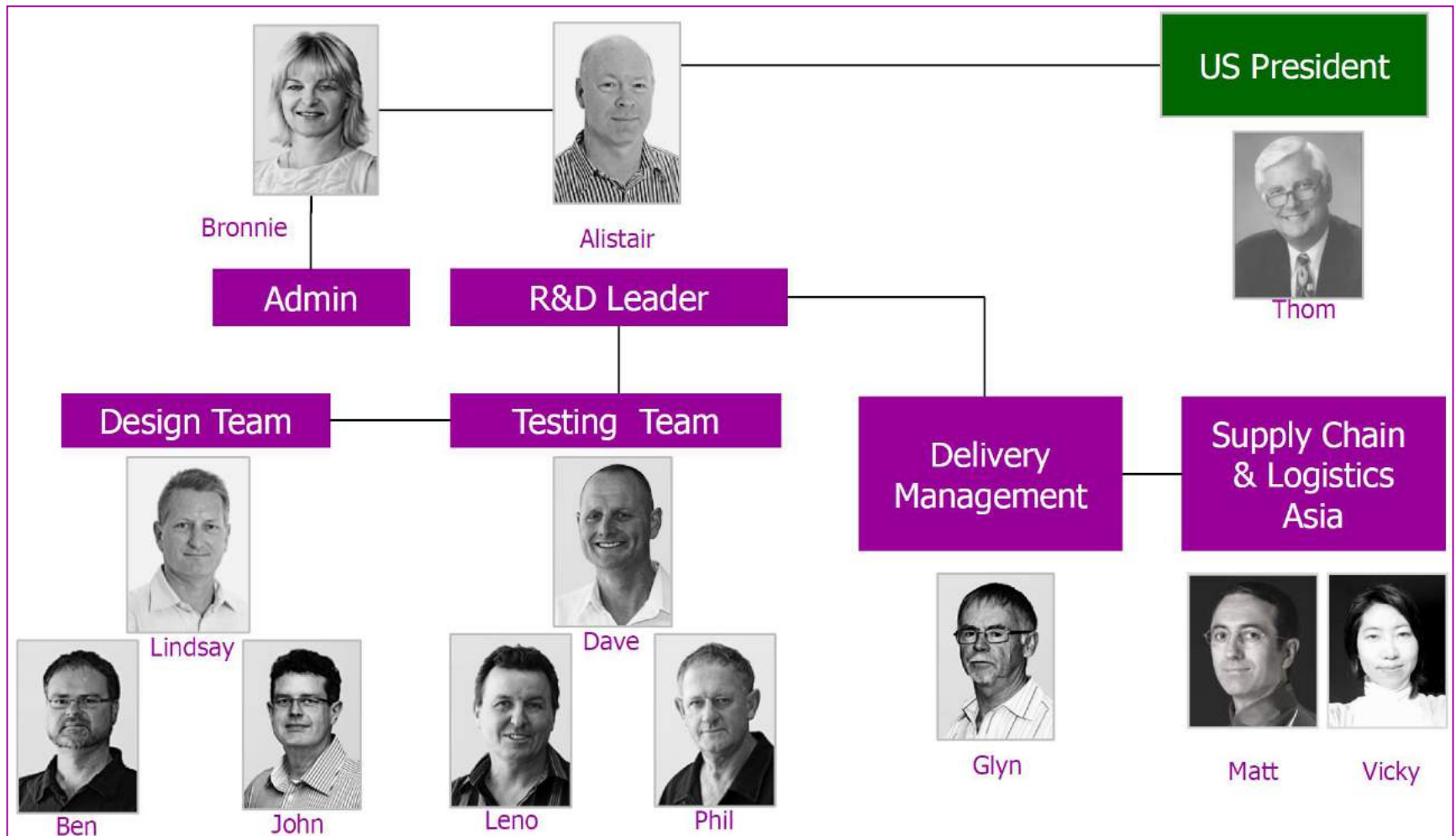
Typical Version 03 Pump, Exploded Assembly View



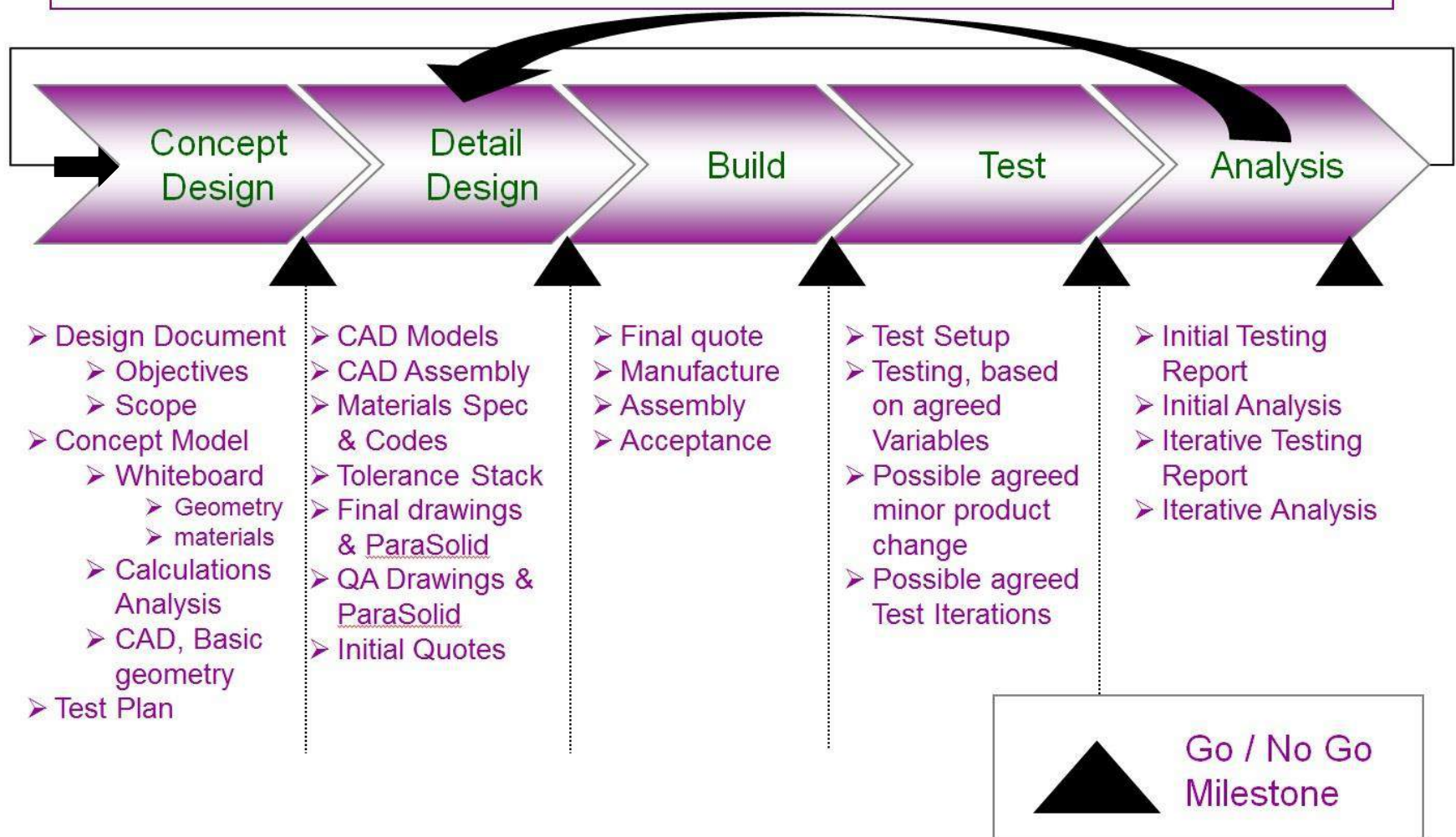
- A** Body Parts
- B** Shaft / Swash Assembly Parts
- C** Plastic Seals (replaceable parts)



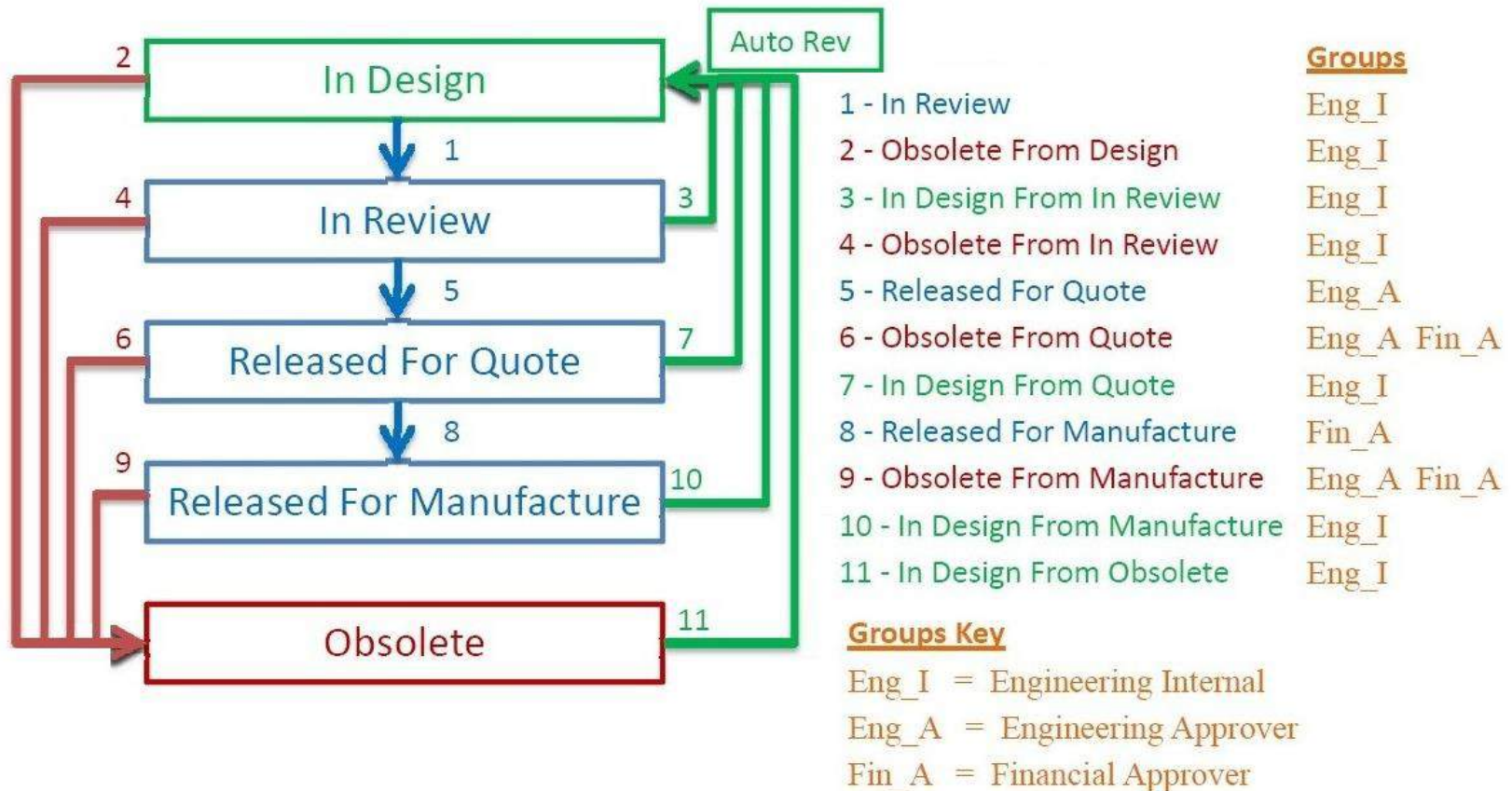
The SwashPump Team (2009)



Typical Development Process

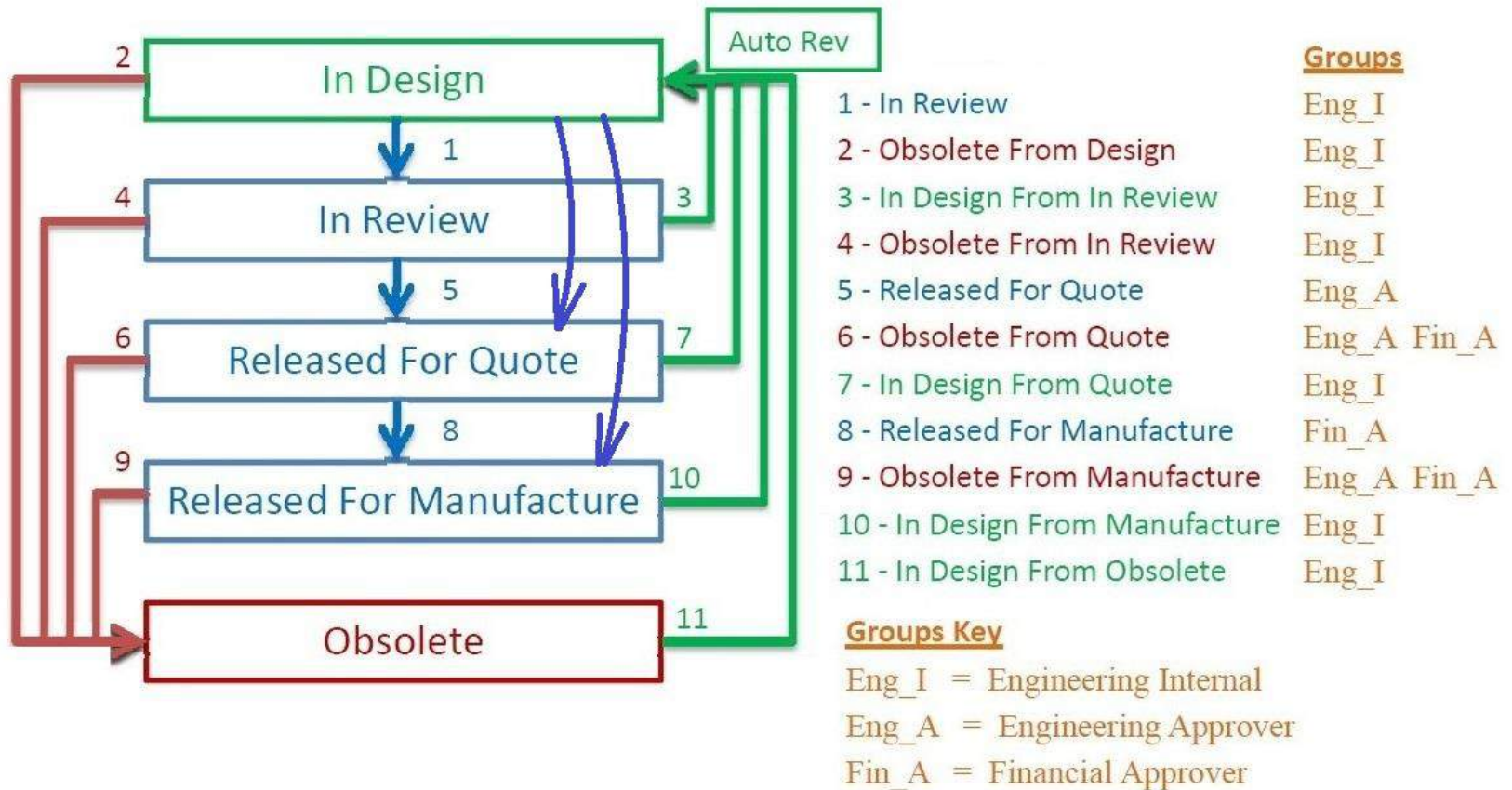


The SwashPump PDMWE workflow



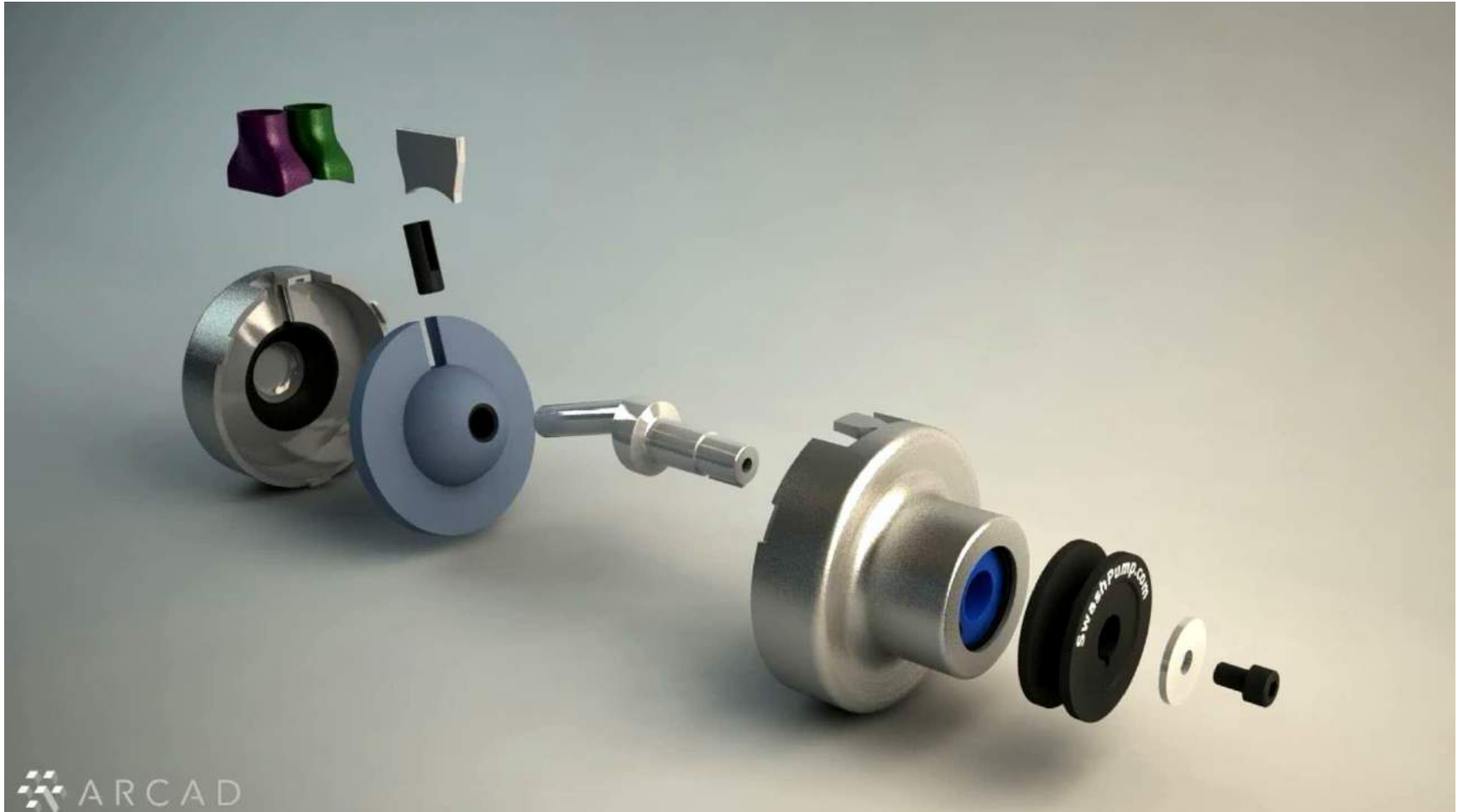
Note revision control of part configurations was done via revision management of drawings (only) each drawing created from specific part configurations.

Improved SwashPump PDMWE workflow

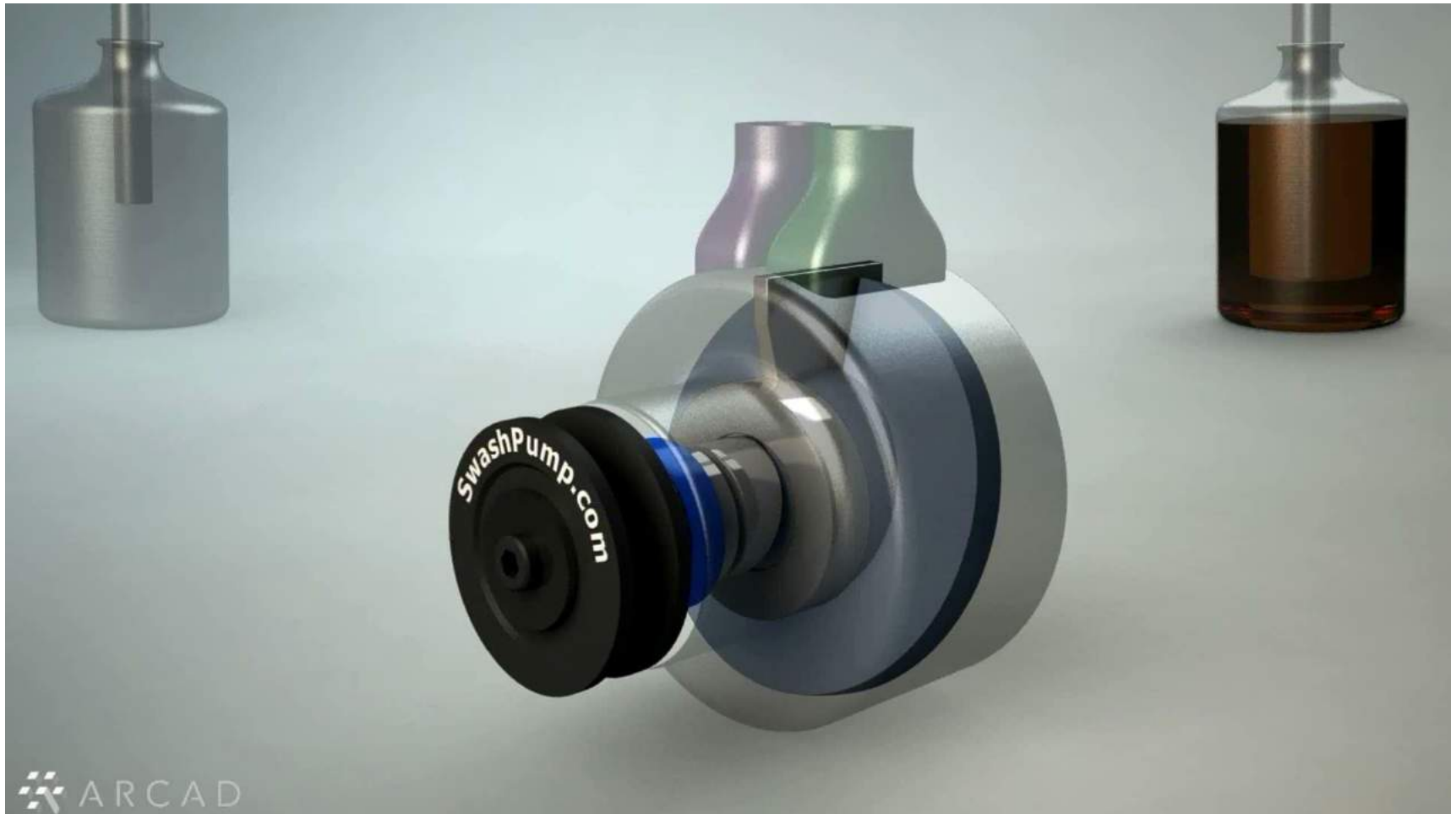


Note revision control of part configurations was done via revision management of drawings (only) each drawing created from specific part configurations.

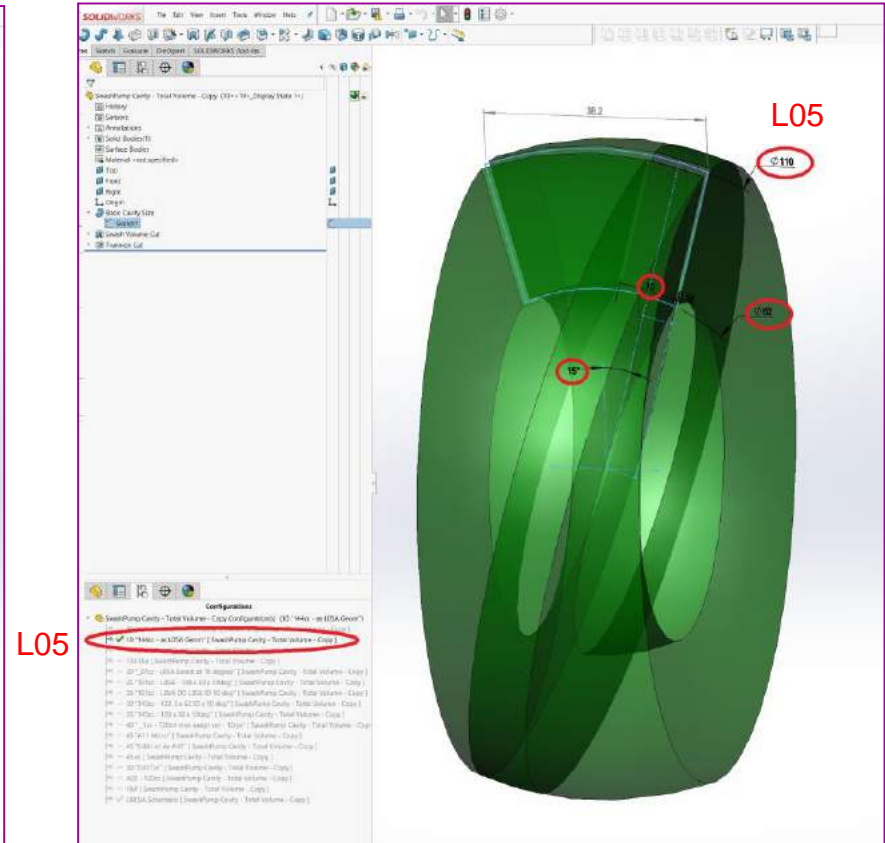
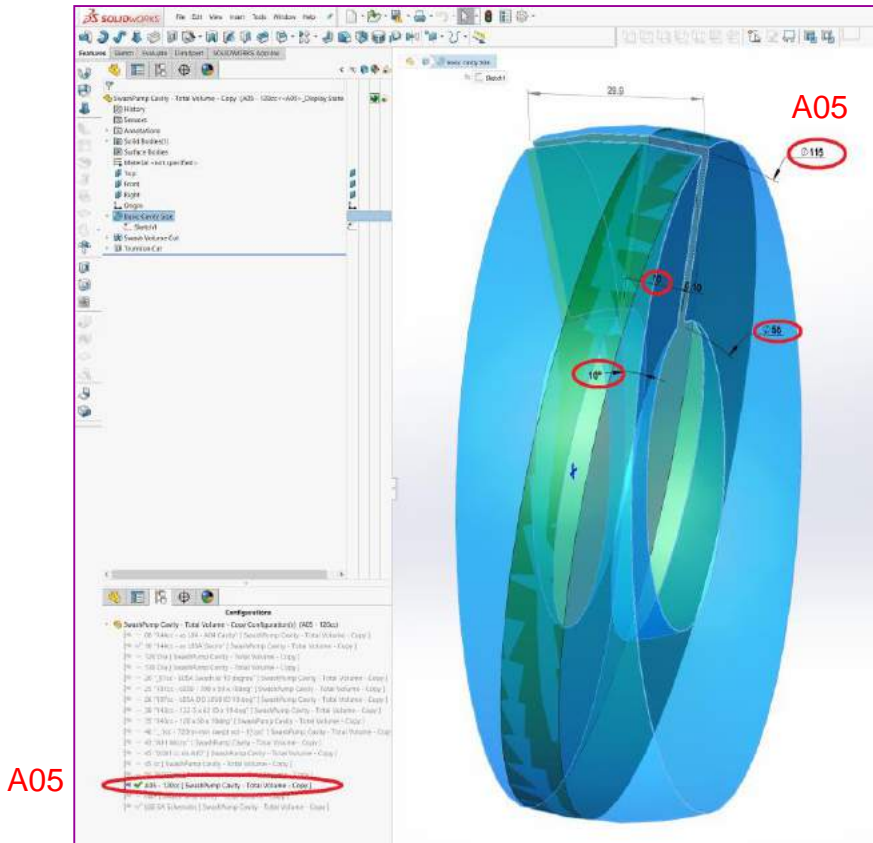
SwashPump Version 05 Liquid Pump Assembly



SwashPump Version 05 Liquid Pump Motion



The Swept Volume / Important Dimensions



- 1) Outer Swash Diameter
- 2) Inner Swash Diameter (Hub Diameter)
- 3) The Cone Angle (Angle of Nutation)
- 4) Swash Thickness (subtracted volume)

Nutating Plate Pump (non rotational)

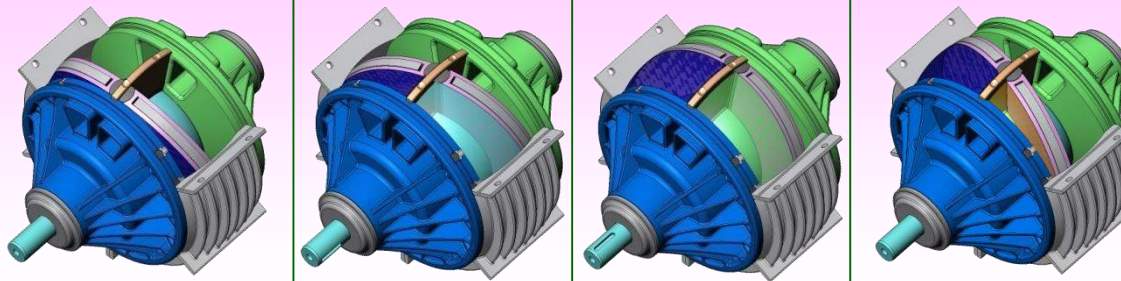


Vacuum Application Swash v Roots

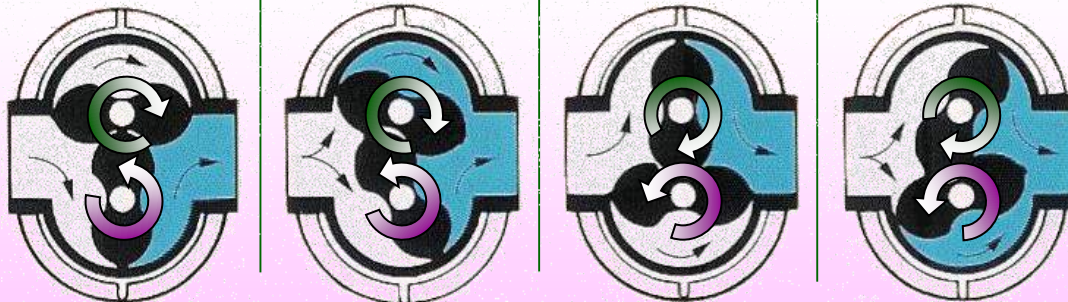
Valves & Mechanical Seals



Valves (Reeds) can be fitted at the exhaust ports



Cant Valve Roots Blower, suffers reflux positions 2 & 4



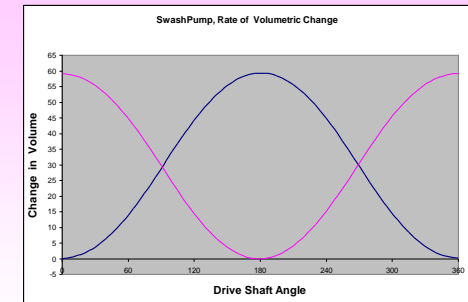
Position 1

Position 2

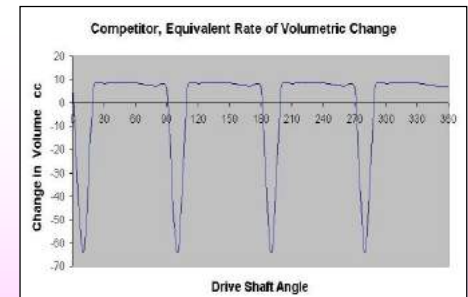
Position 3

Position 4

Swash Pump Flow



Roots Pump Flow



1500
NI/min

Swash
Pump

Current
Technology

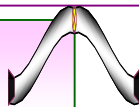
Volume

V

V

2

**Shock Free
Compression
(sinusoidal)**

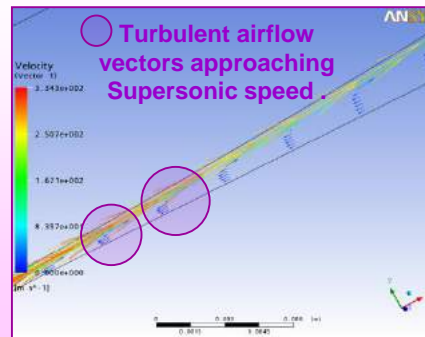
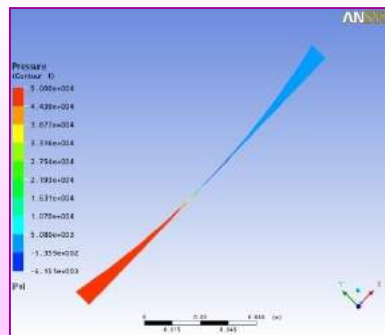
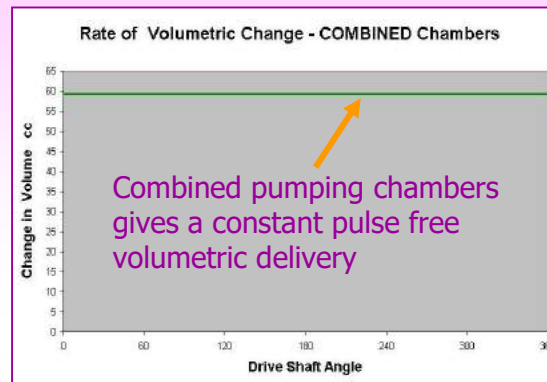
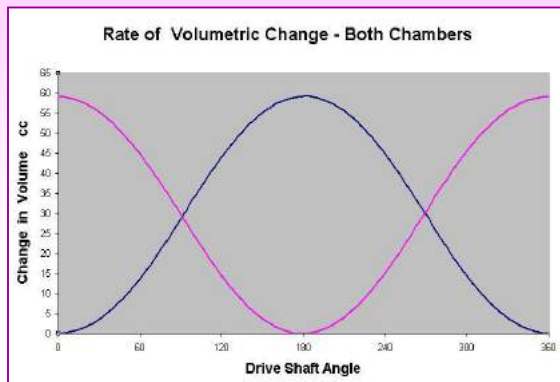


Flow & Leakage

Low Leakage
Low Thermals



The Smooth Shock free sinusoidal compression from both Swash pumping chambers combine as shown



- ➡ The combined smooth shock free compression of the Swash Pump gives a constant flow beyond the valves
- ➡ CFD modelling of the Swash to Cone air leakage velocity shown, lower right hand image.

1500 NI/min	Swash Pump	Roots Pump
Volume	V	V
Thermals	T	1.4 T

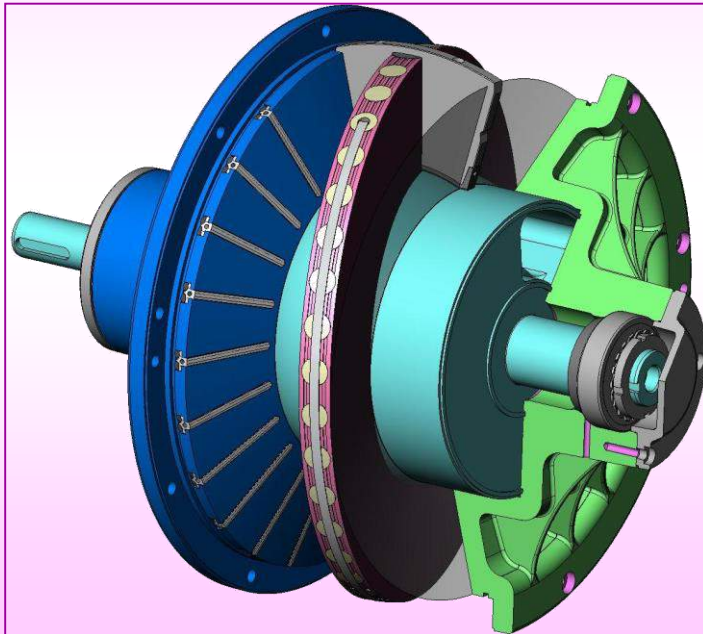


Mechanical Seals Sealing Vanes

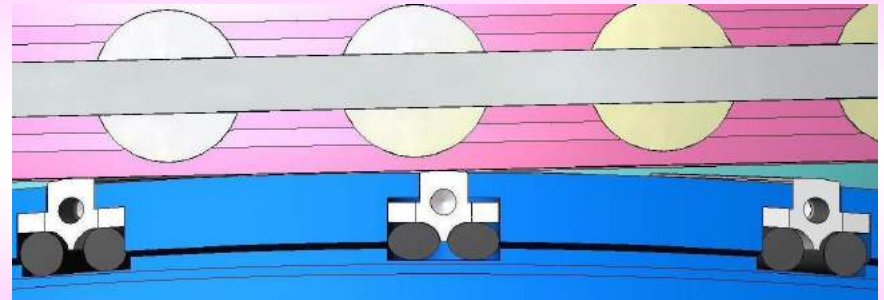
Mechanical
Seals



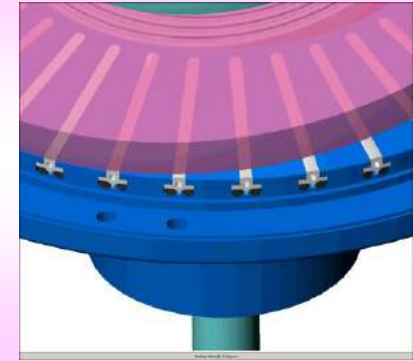
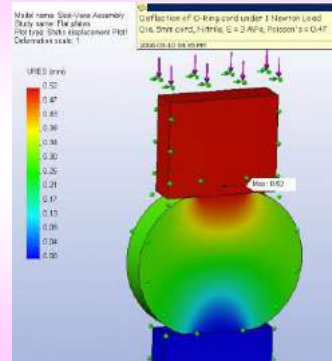
- ➔ The Sealing Vanes were introduced to ascertain what performance improvement would be achieved.
- ➔ Cost / benefit was marginal.



Sealing Vane End View



For AVI Video, Double Click Bottom right Image



The Swash Sealing Ring

An Important Mechanical Seal for pumping air / gas

Mechanical
Seals

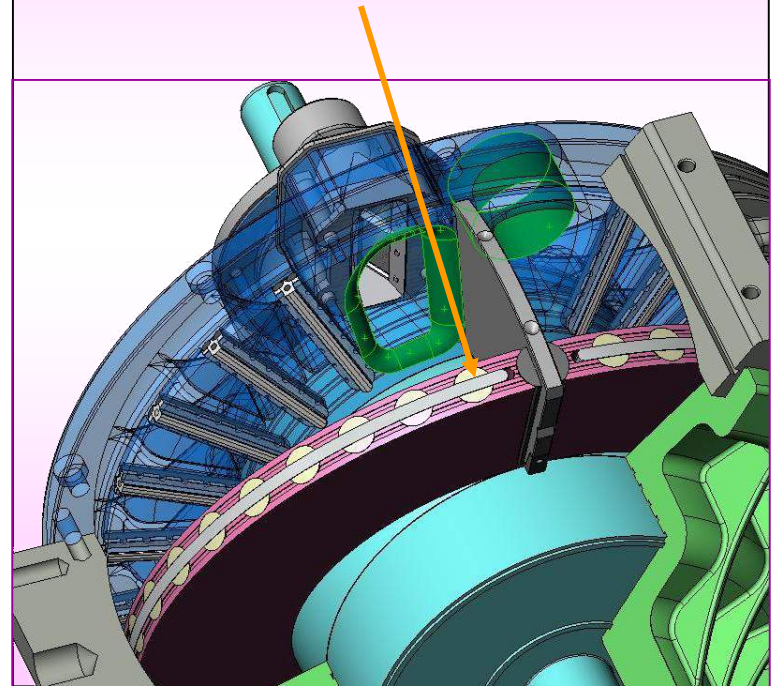


Sealing Ring

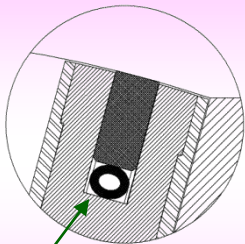


Version 03 Pump Design

The Sealing Ring
has a 350° arc



X-Sec & video, double click picture



The Sealing Ring is
Energized with a
Rubber Backing Spring



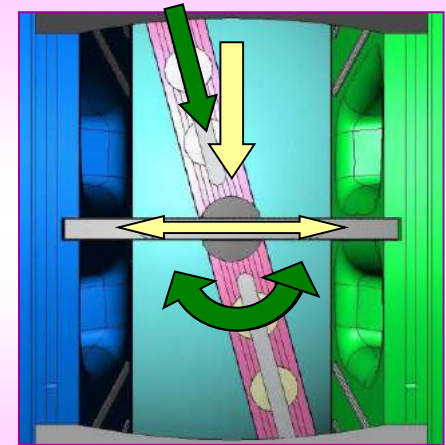
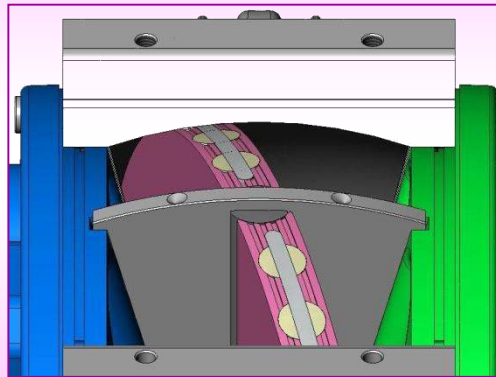
The Trunnion Bearing

Friction surfaces	Load type	Distance / cycle	Average Velocity
Trunnion to Divider	Perpendicular	180mm	Average 3.5m/sec
Swash to Trunnion	Radial	12mm	0.23m/sec

Trunnion Forces

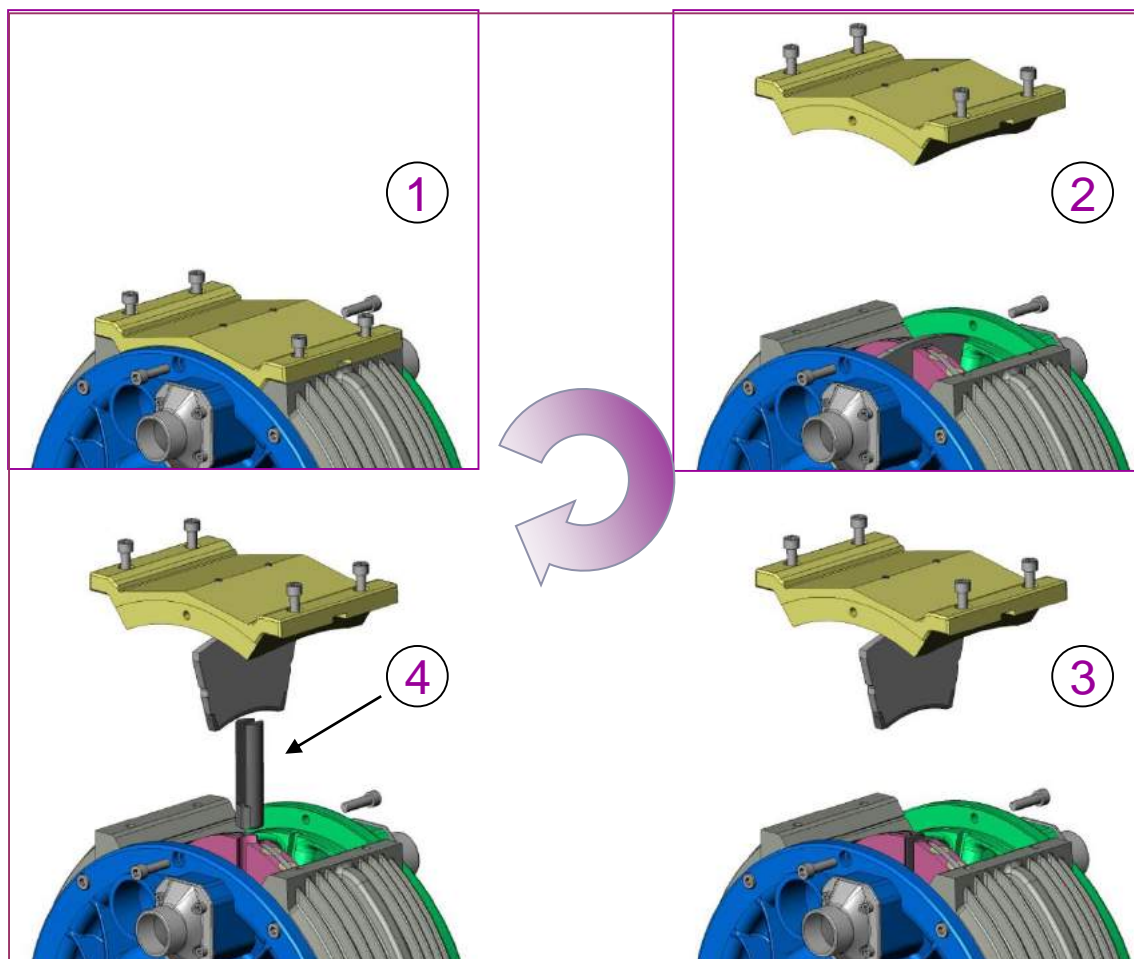


The Trunnion acts as both a Seal & Bearing as illustrated



Trunnion Serviceability

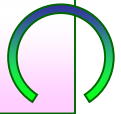
1. Loosen 6 Fasteners
2. Remove Divider Retainer (Body Top)
3. Remove Divider & its attached Seal
4. Remove Trunnion from Swash Plate
5. Replace Trunnion with new item if required, Reverse Steps 1 to 4.



Exhaust Reed Valves

Required for higher than **300mbar Vacuum** applications

Mechanical
Seals

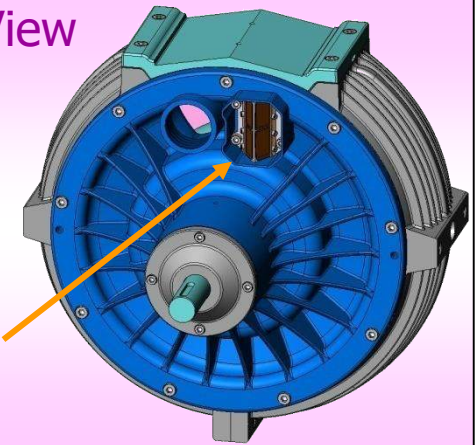


➡ Carbon fibre Reed valves



03 Pump External View

Exhaust Valve Enclosure
integrated on Cone Plates



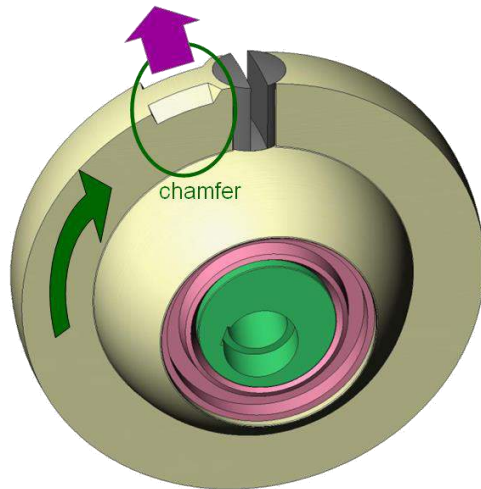
Reed valve in action



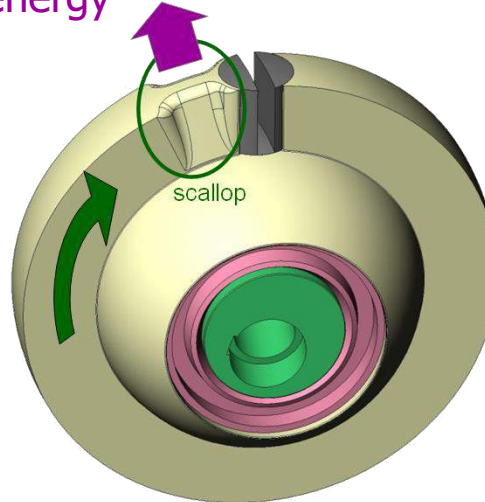
L04 Liquid Pump, Exhaust Analysis

⇒ Swash Plate Liquid Pump geometry options analyzed in exhaust region.

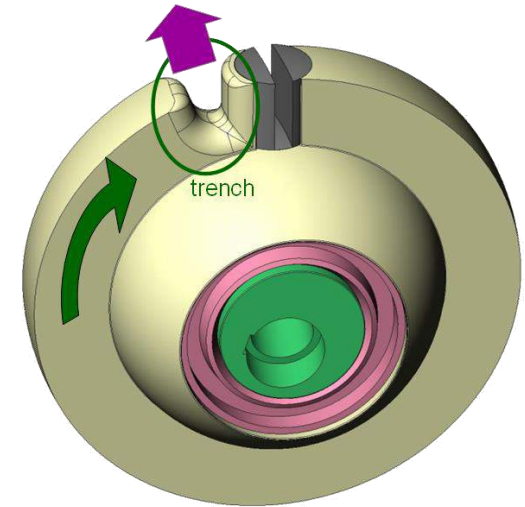
- ⇒ Reduced peak fluid velocity
- ⇒ Reduced fluid kinetic energy
- ⇒ Reduced max static pressures



Chamfer



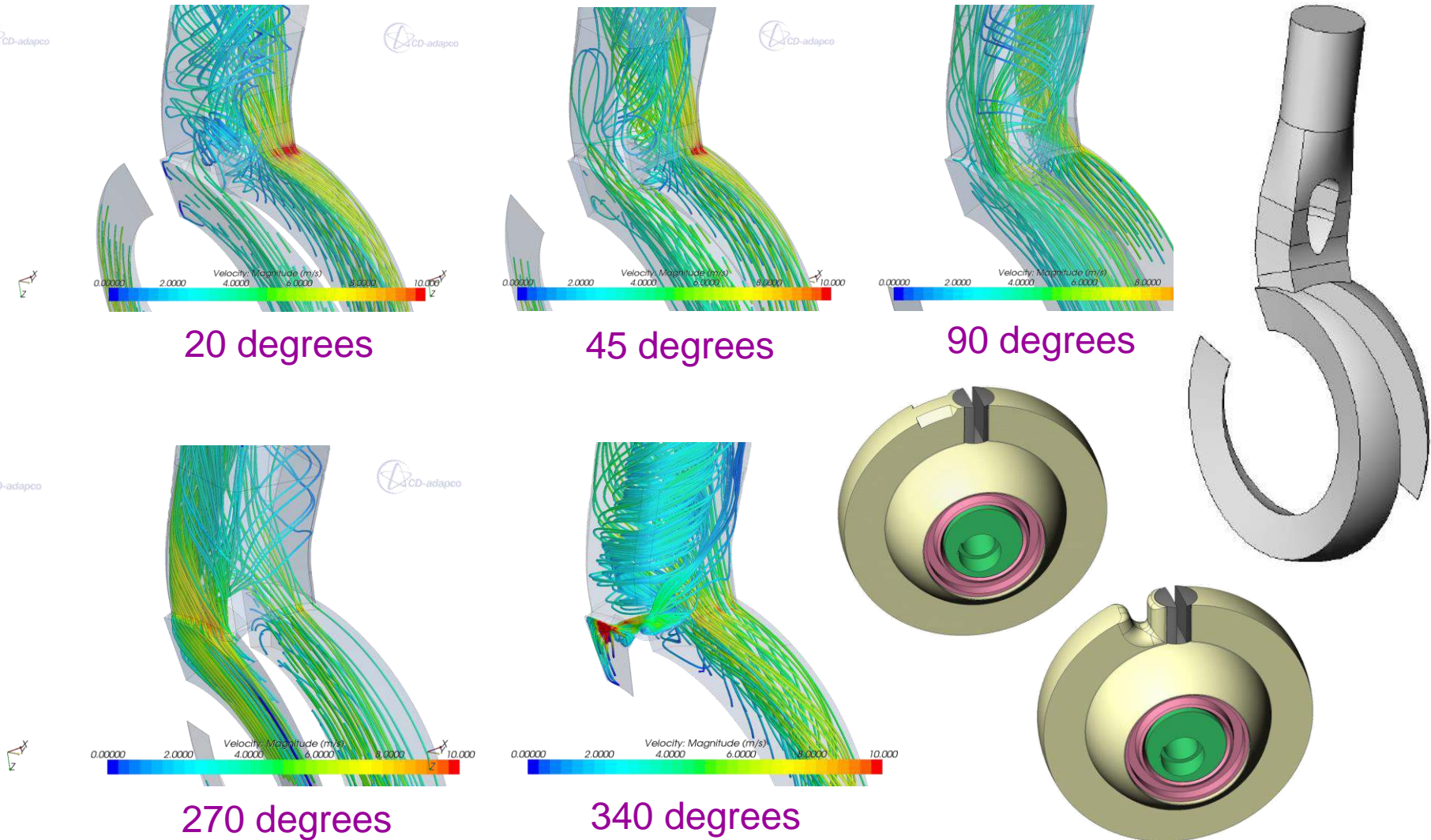
Scallop



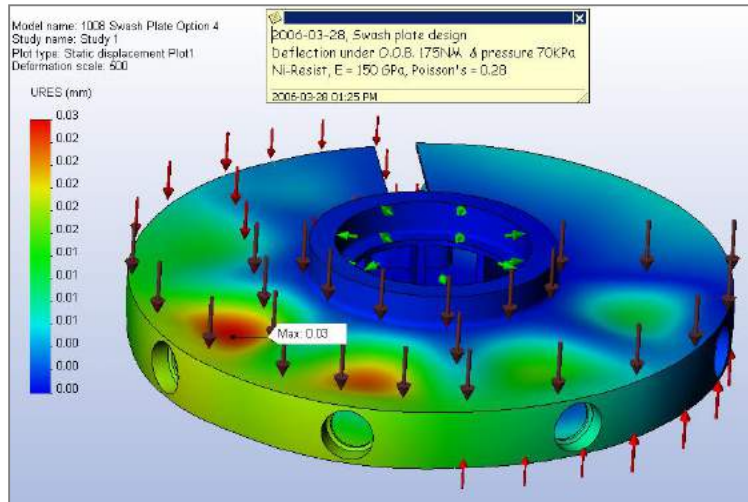
Trench

- ⇒ Green arrow = direction of fluid during compression cycle
- ⇒ Purple arrow = direction of fluid discharge through port

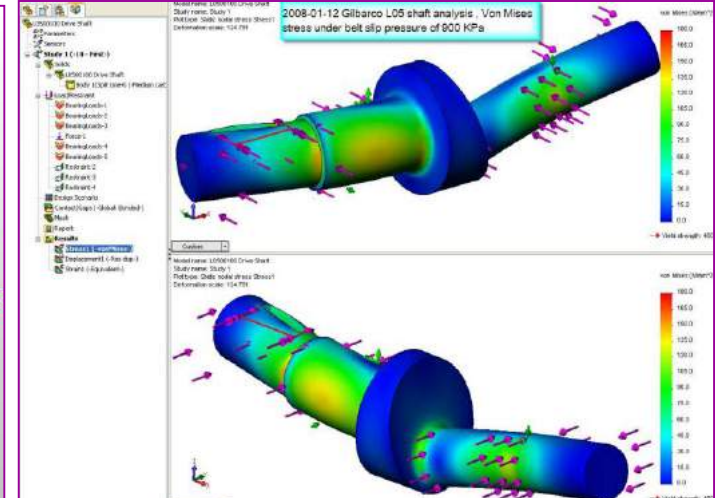
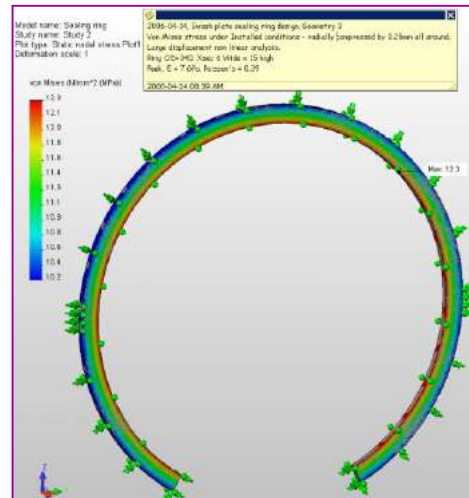
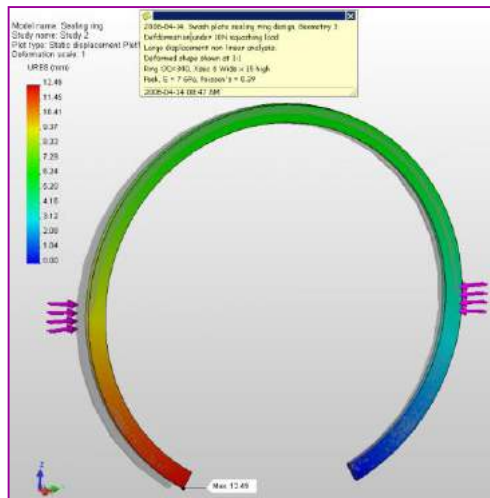
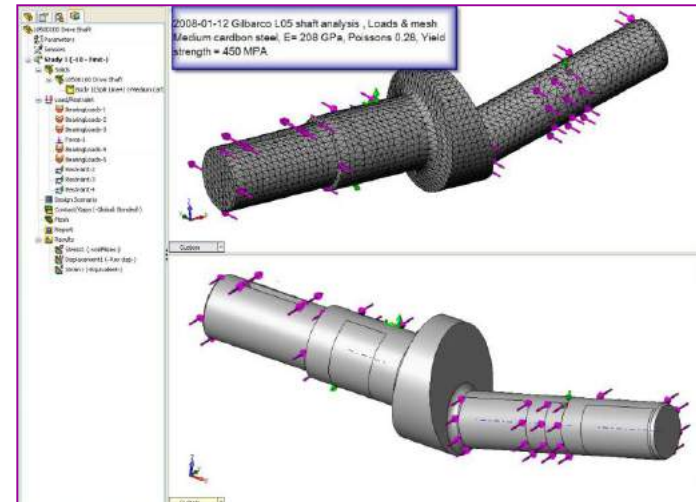
L04 Liquid Pump, CFD Exhaust Analysis



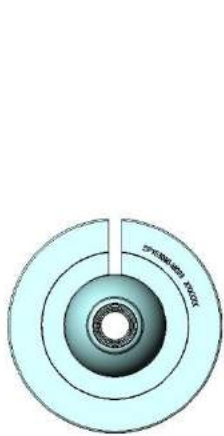
SolidWorks Simulation, where required



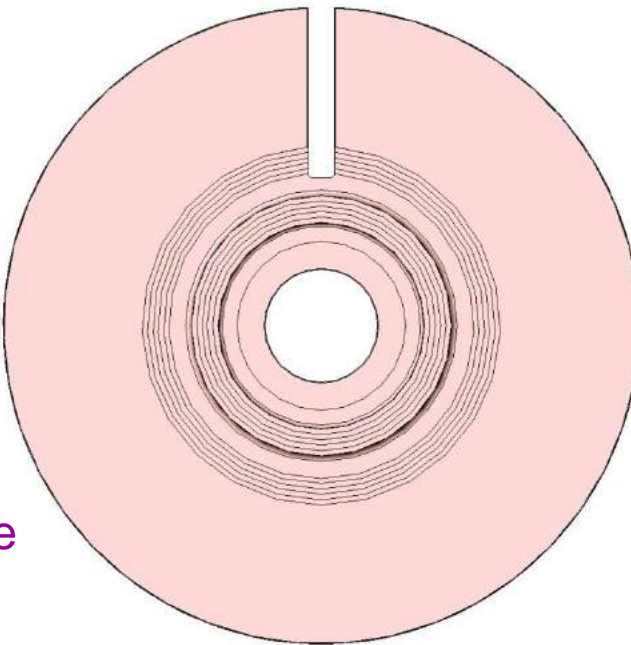
Simulation of Hollow Swash shown above



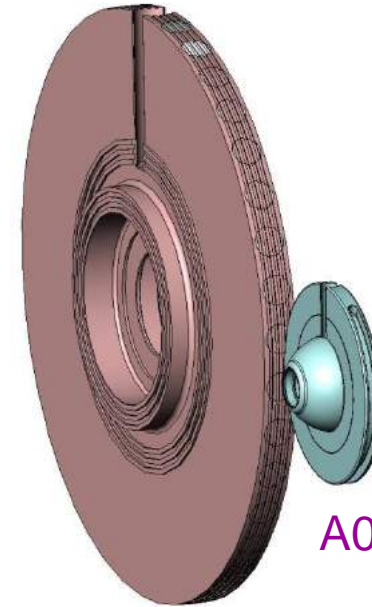
Dynamic balance of a Nutating (Wobble) Plate



A05-V4
Swash Plate



Series 03 Swash Plate



A05 = 10° Slant

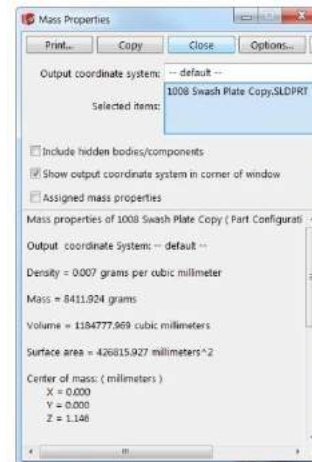
Series 03 = 15° Slant

Size does matter! & Slant (travel) angle

Dynamic balance of a Nutating Plate



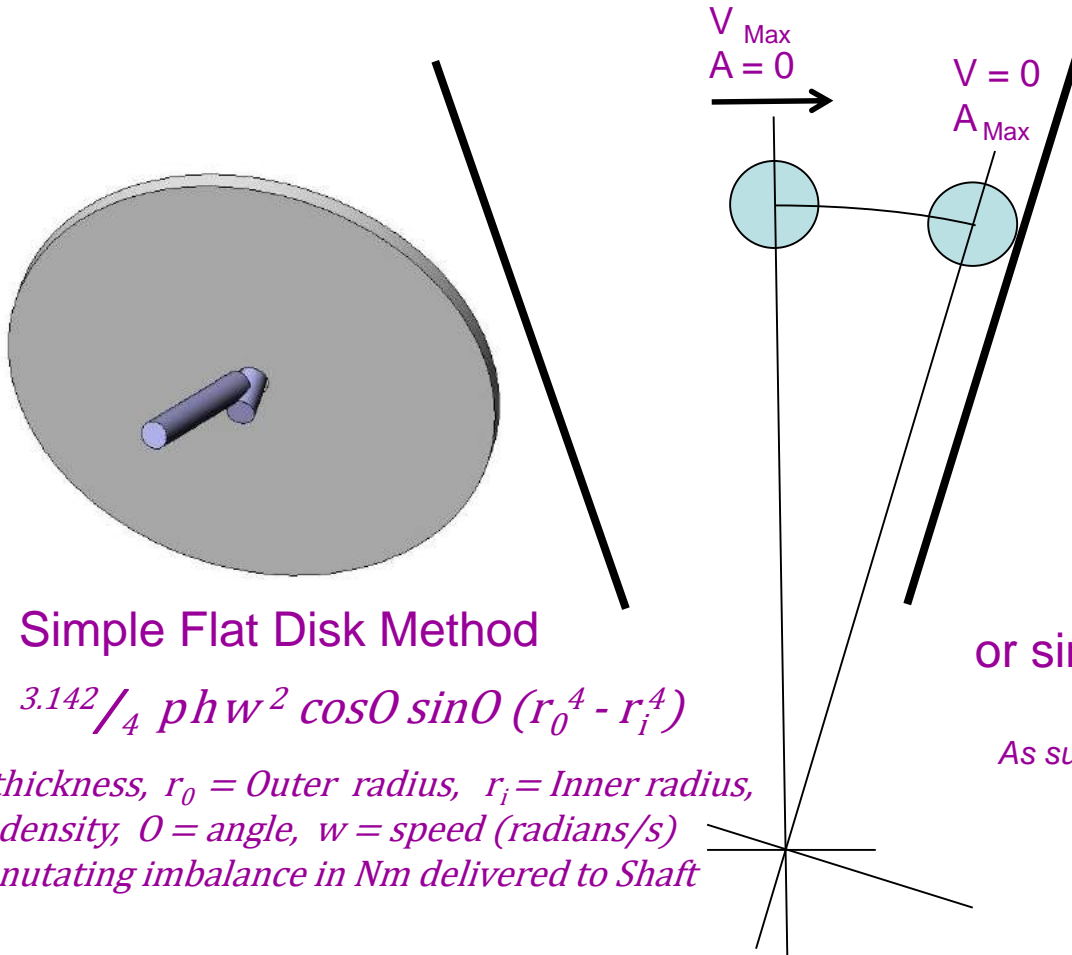
Series 03 Swash Plate



Weight / Mass Reduction

So obviously Mass / Weight matters!

Dynamic imbalance calculation methods



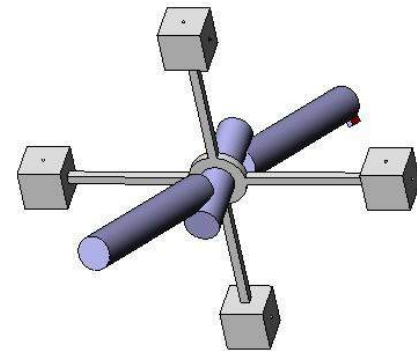
Simple Flat Disk Method

$$C = \frac{3.142}{4} p h w^2 \cos O \sin O (r_o^4 - r_i^4)$$

h = thickness, r_o = Outer radius, r_i = Inner radius,

p = density, O = angle, w = speed (radians/s)

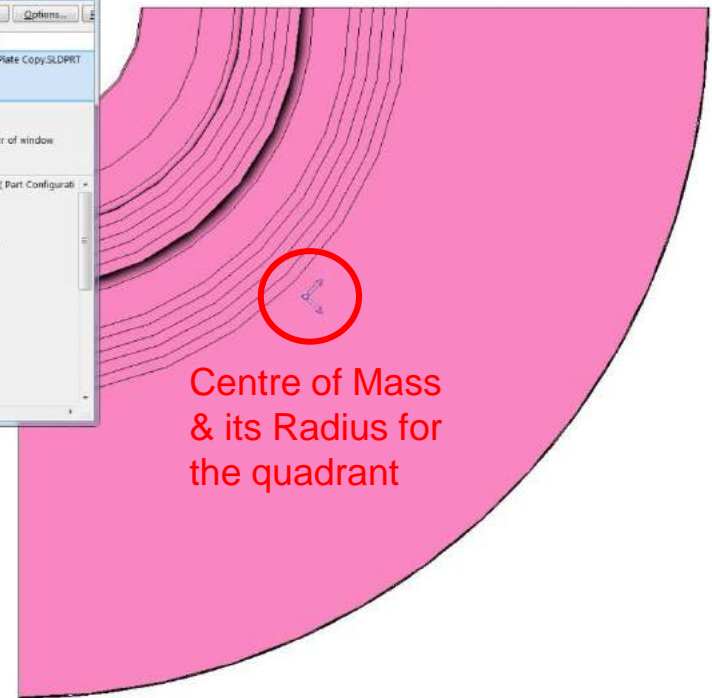
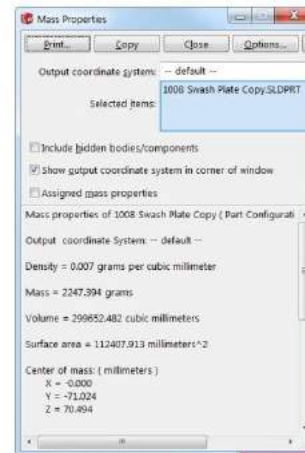
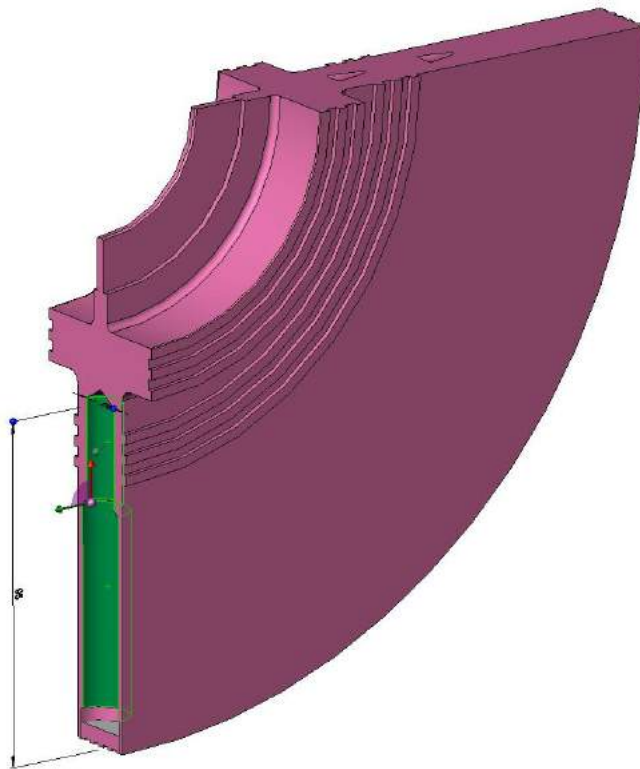
C = nutating imbalance in Nm delivered to Shaft



or simple 4 Lump Mass Method

A close approximation method,
As suggested by Lan from IRL see next slides

Dynamic balance of Large 03 Plate



Dynamic balance of Large 03 Plate

➤ 4 Lump Mass (Close Approximation) Method

$\frac{1}{4}$ Mass of Swash = 2.13 kg

effective radius $r = 0.10$ m

Distance $d = r \times \sin 15^\circ = 0.0259$ m

Speed (1500rpm) $\omega = 157$ rad/s

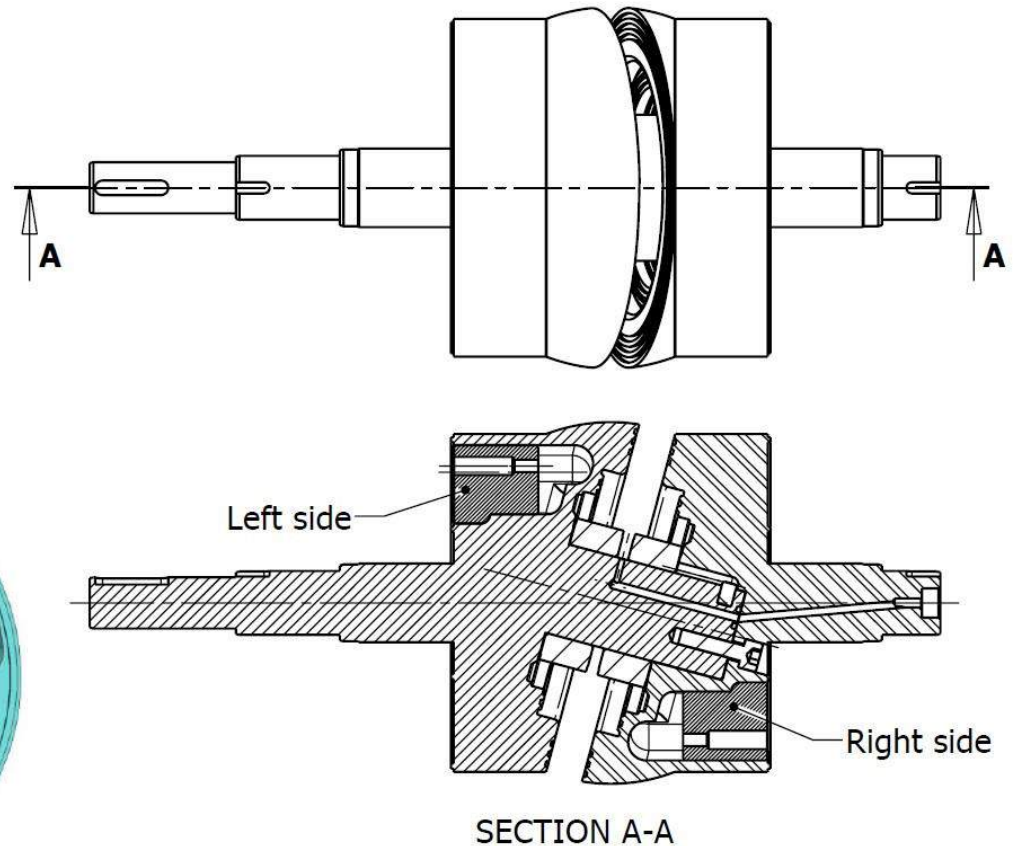
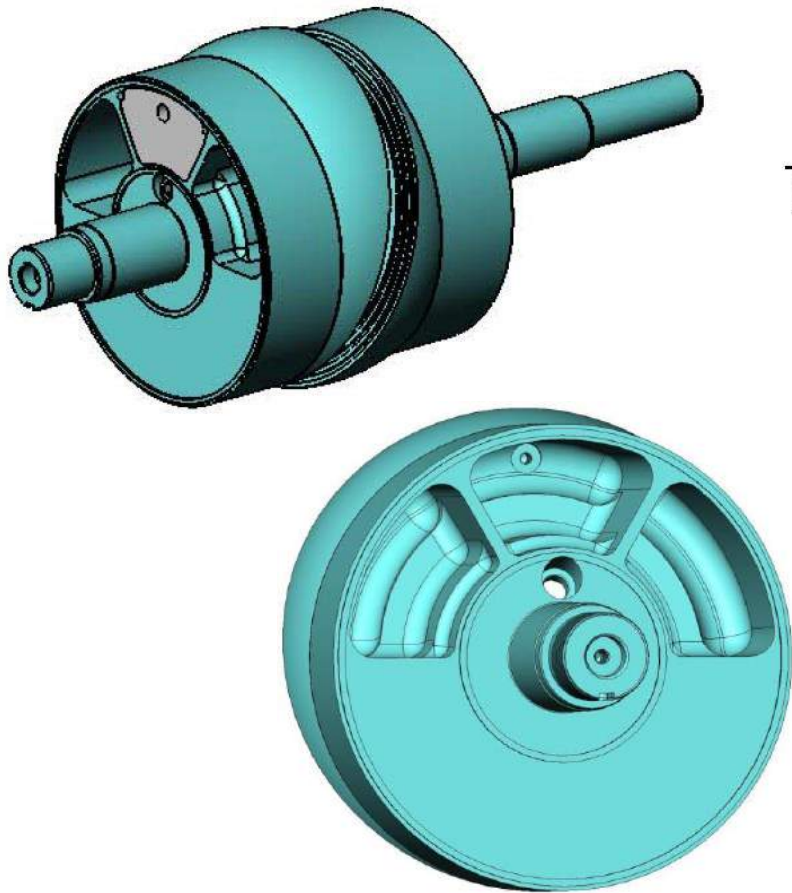
Acceleration = $d \omega^2 = 0.0259 \times 157^2 = 638$ m/s²

$F = m a = 2.13 \times 638 = 1358.9$ N

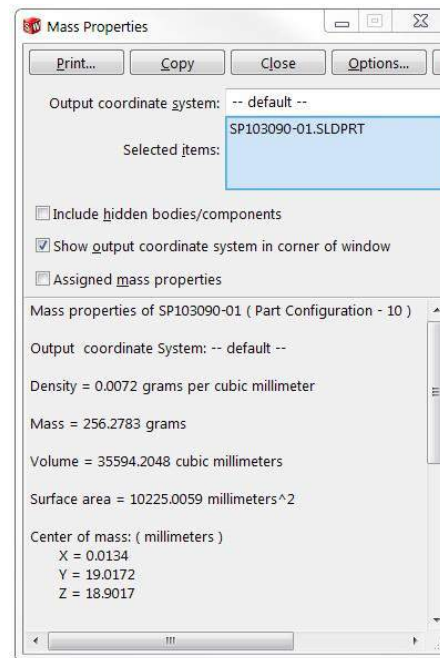
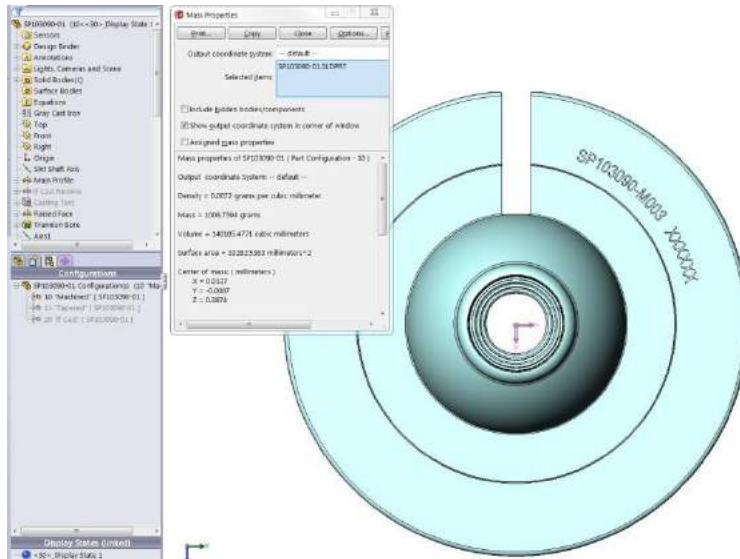
Nutating Imbalance Couple = $2 r F = 2 \times 0.1 \times 1358.9 = 272$ Nm

➤ **This was a significant imbalance for the total mass of 03 pump so we compensated for this within the pump.**

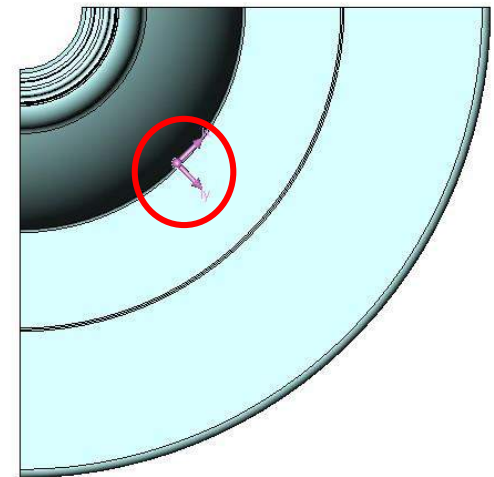
Dynamic balance rectification of large 03 Pump



Dynamic balance of the small A05-V4 Plate



Centre of Mass
& its Radius for
the quadrant



Dynamic balance of A05-V4 Plate

➤ 4 Lump Mass (Close Approximation) Method

$\frac{1}{4}$ Mass of Swash = 0.256 kg

effective radius $r = 0.027$ m

Distance $d = r \times \sin 10^\circ = 0.0047$ m

Speed (1500rpm) $\omega = 157$ rad/s

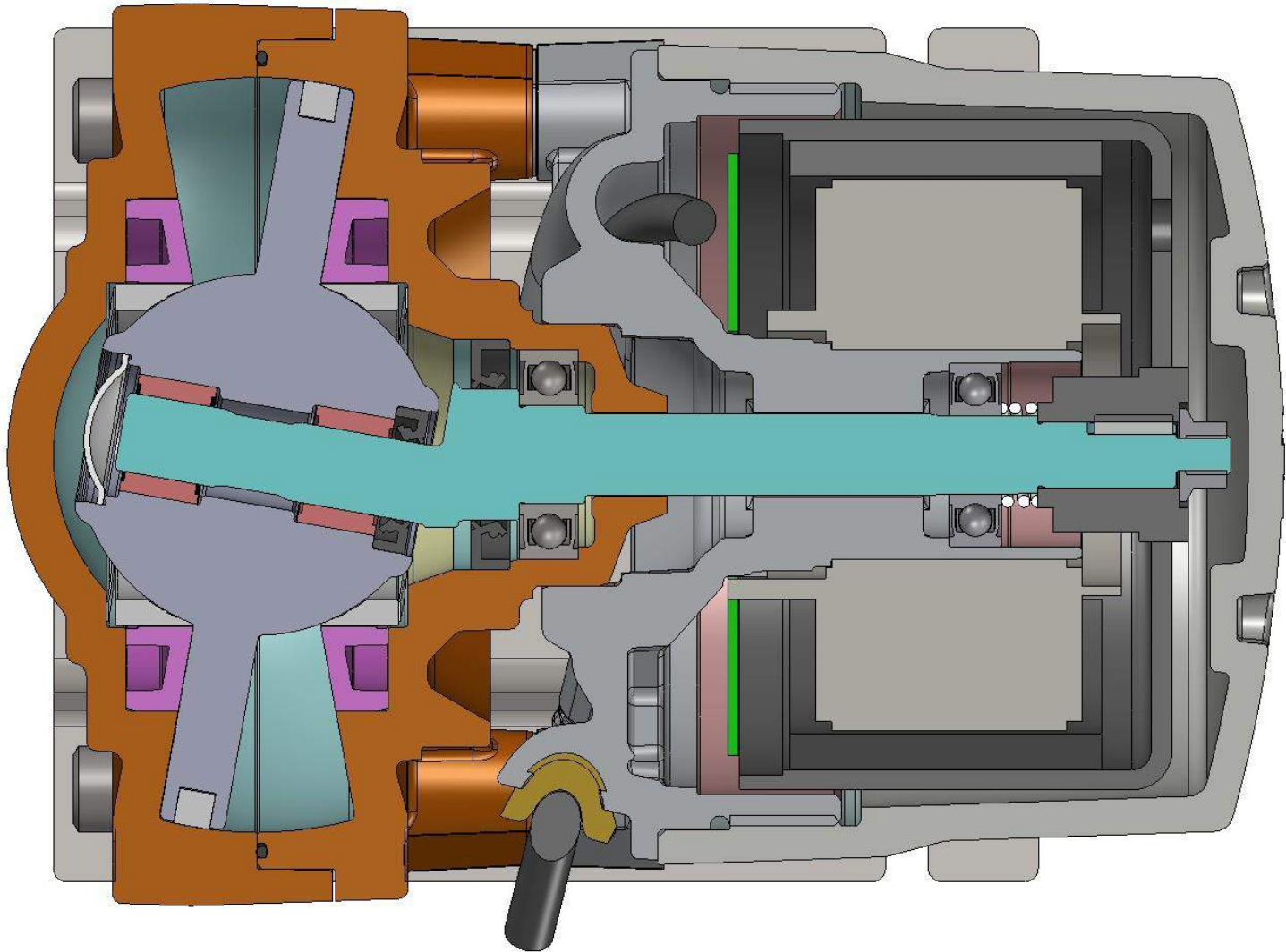
Acceleration = $d \omega^2 = 0.0047 \times 157^2 = 116$ m/s²

$F = m a = 0.256 \times 116 = 29.7$ N

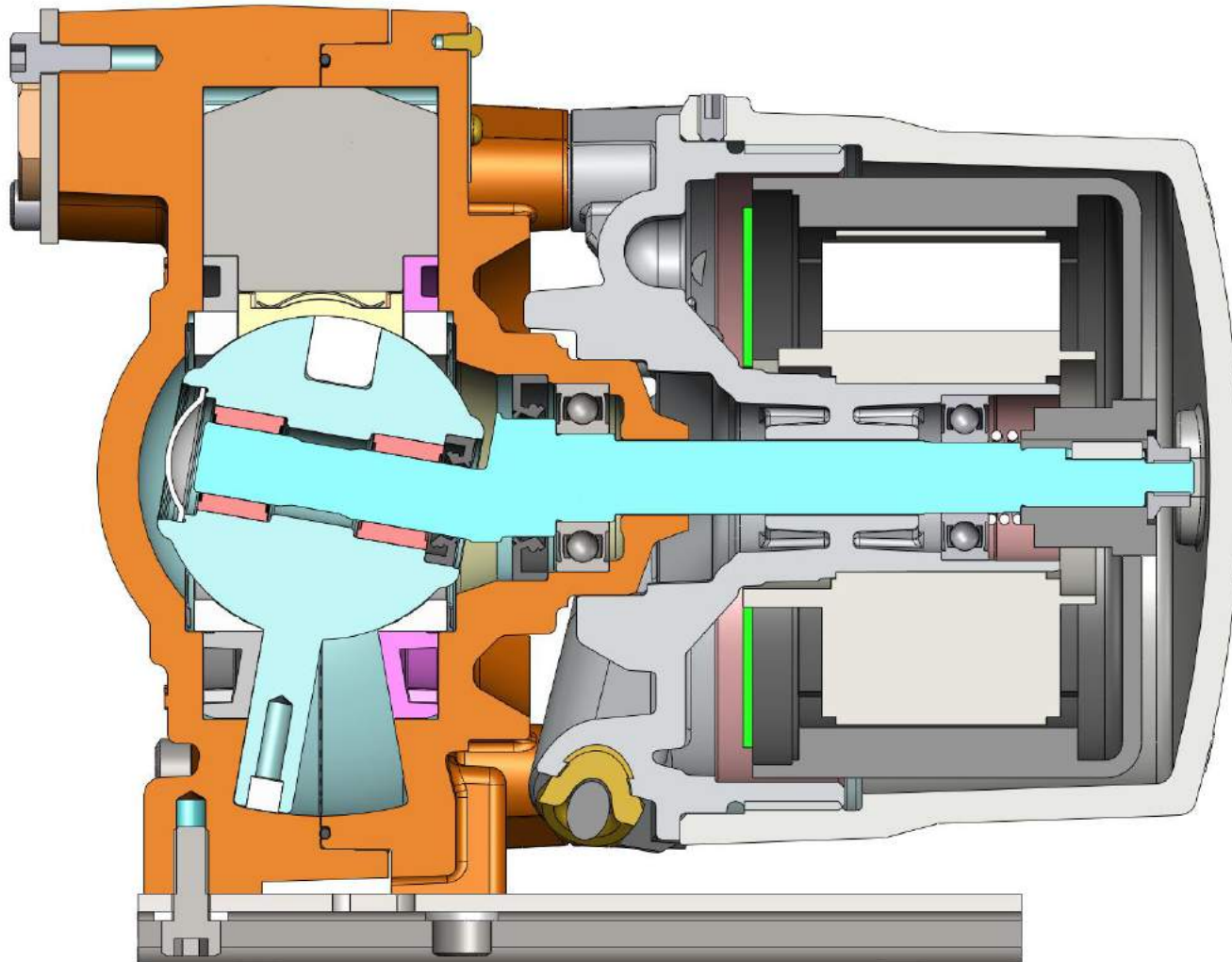
Nutating Imbalance Couple = $2 r F = 2 \times 0.027 \times 29.7 = 1.6 \text{ Nm}$

➤ **This is not a significant imbalance for the total mass of A05 pump.**

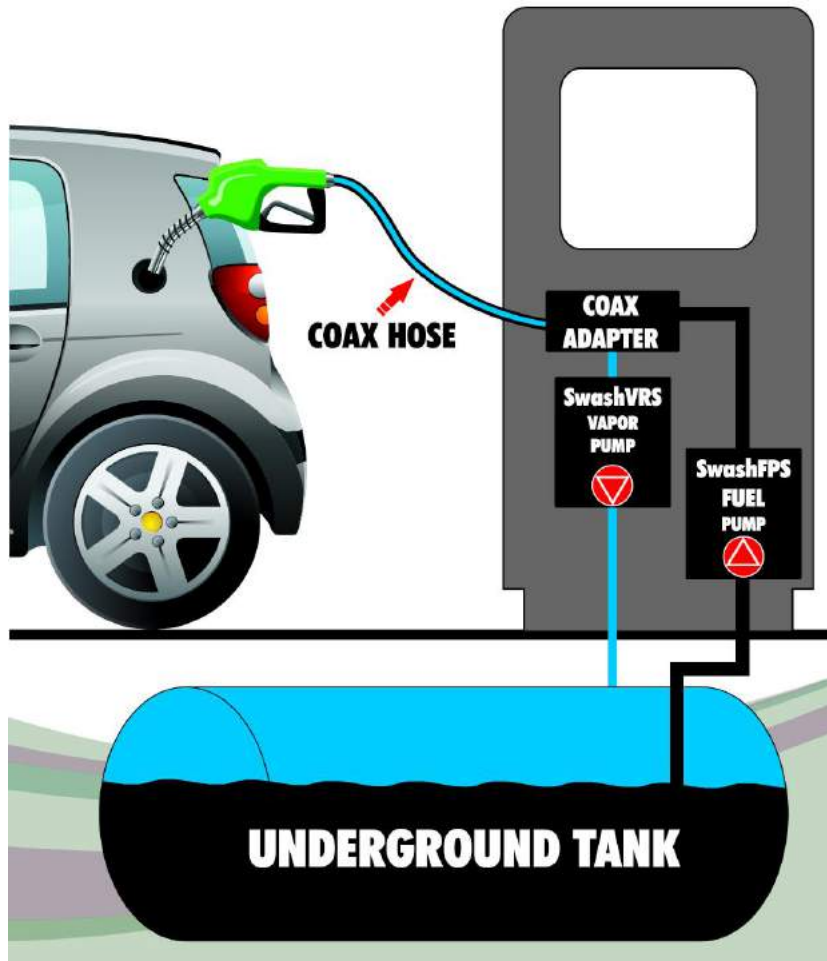
A05-V4 SwashPump Horizontal X-Section



A05-V4 SwashPump Vertical X-Section

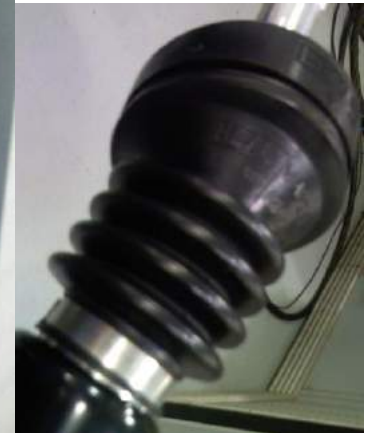


A05-V4 SwashPump Petrol Vapour Recovery

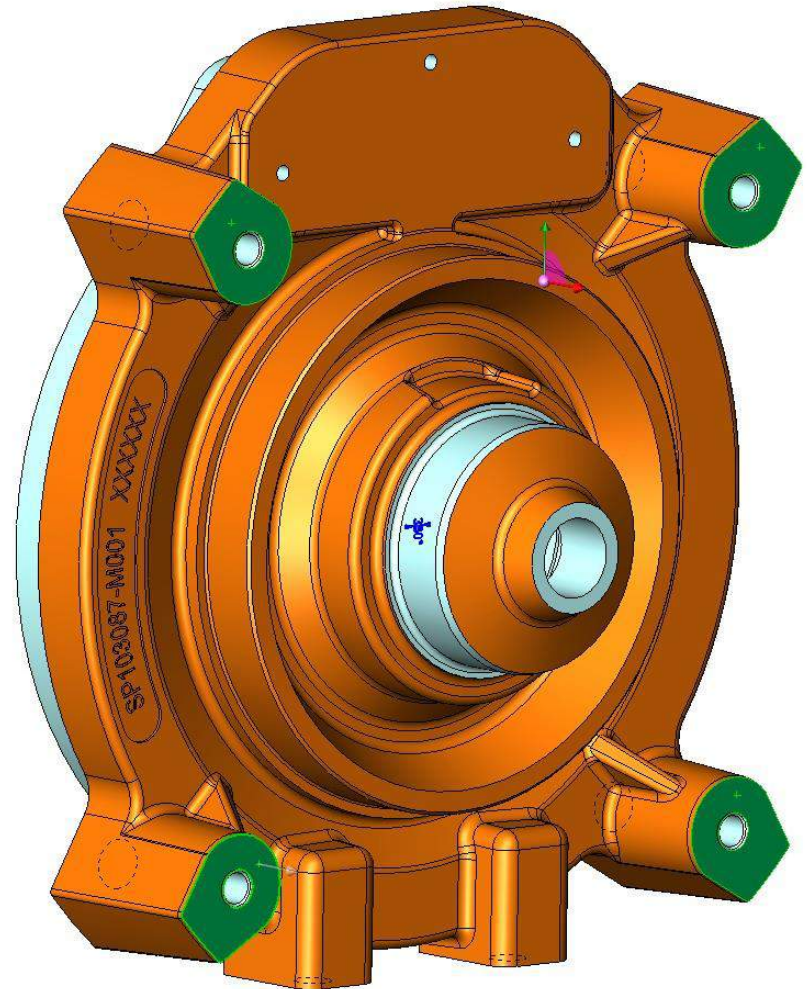
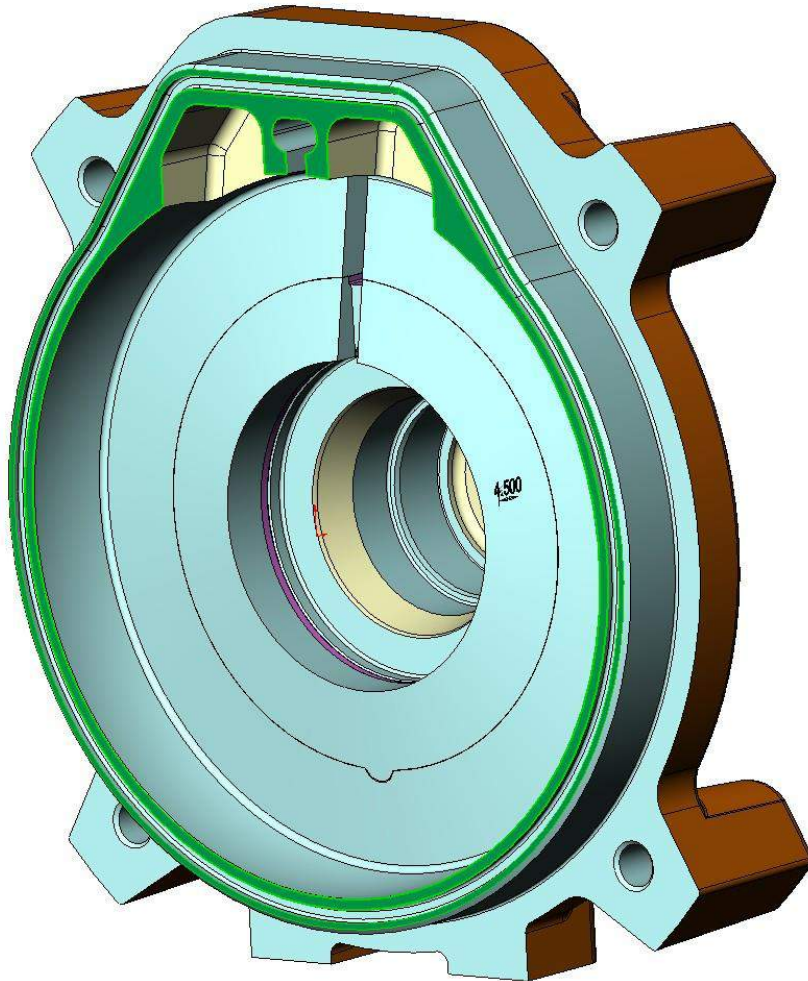


California has been using active Vapour recovery since the 1970's

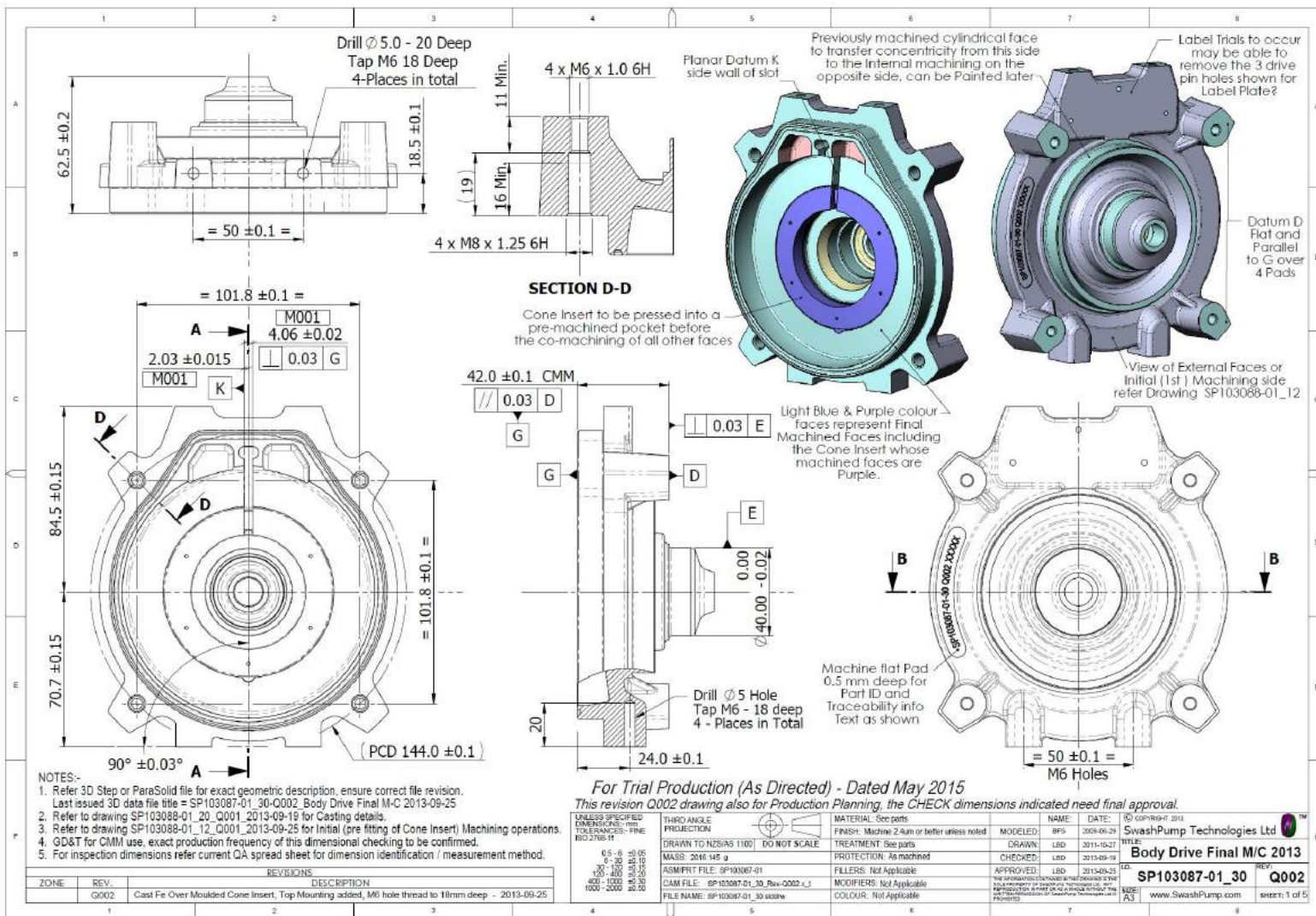
A Rubber boot around the nozzle helps capture the expelled fumes



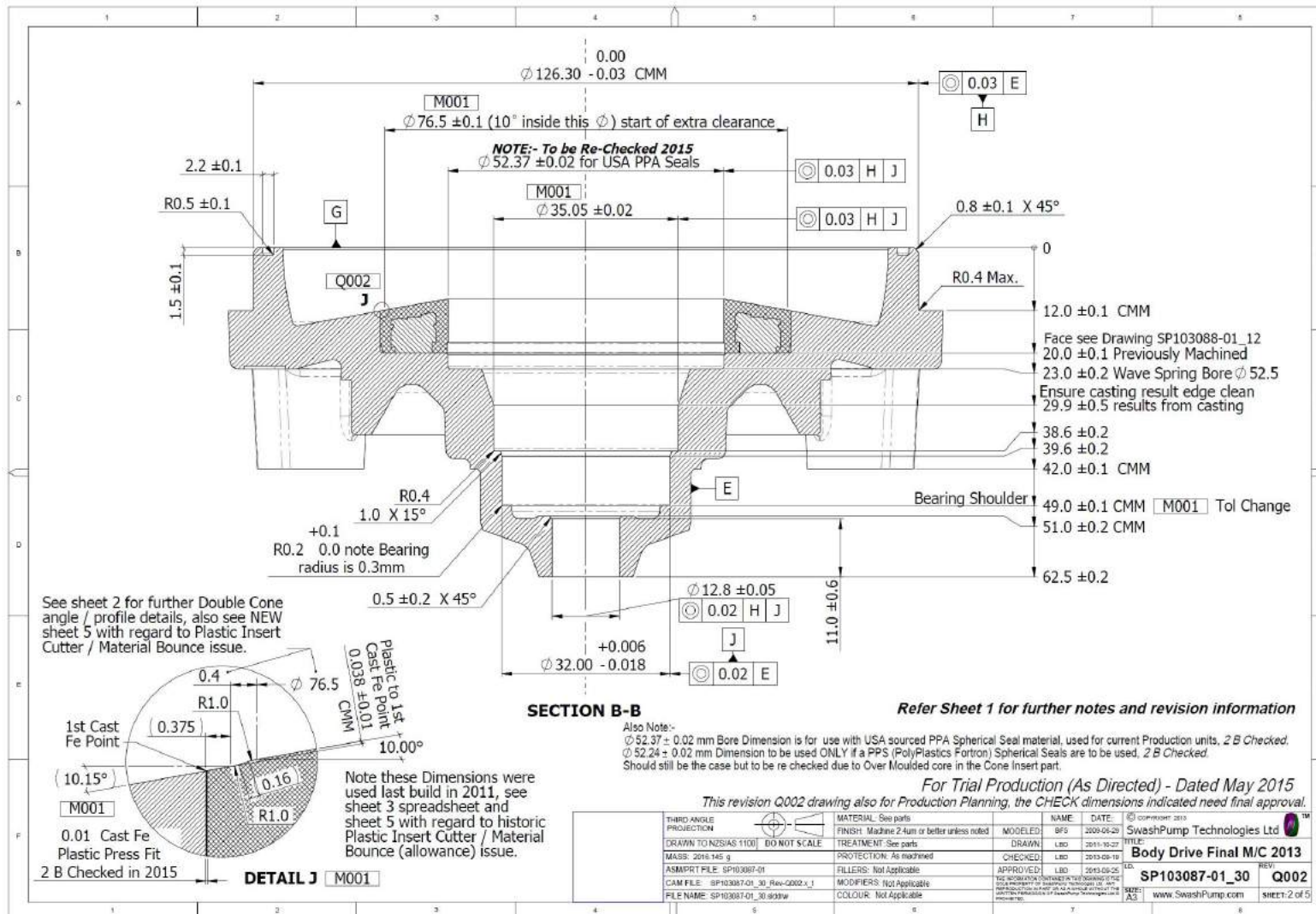
QA - Body Drive Motor Mount, Parallel faces



Drawings & QA

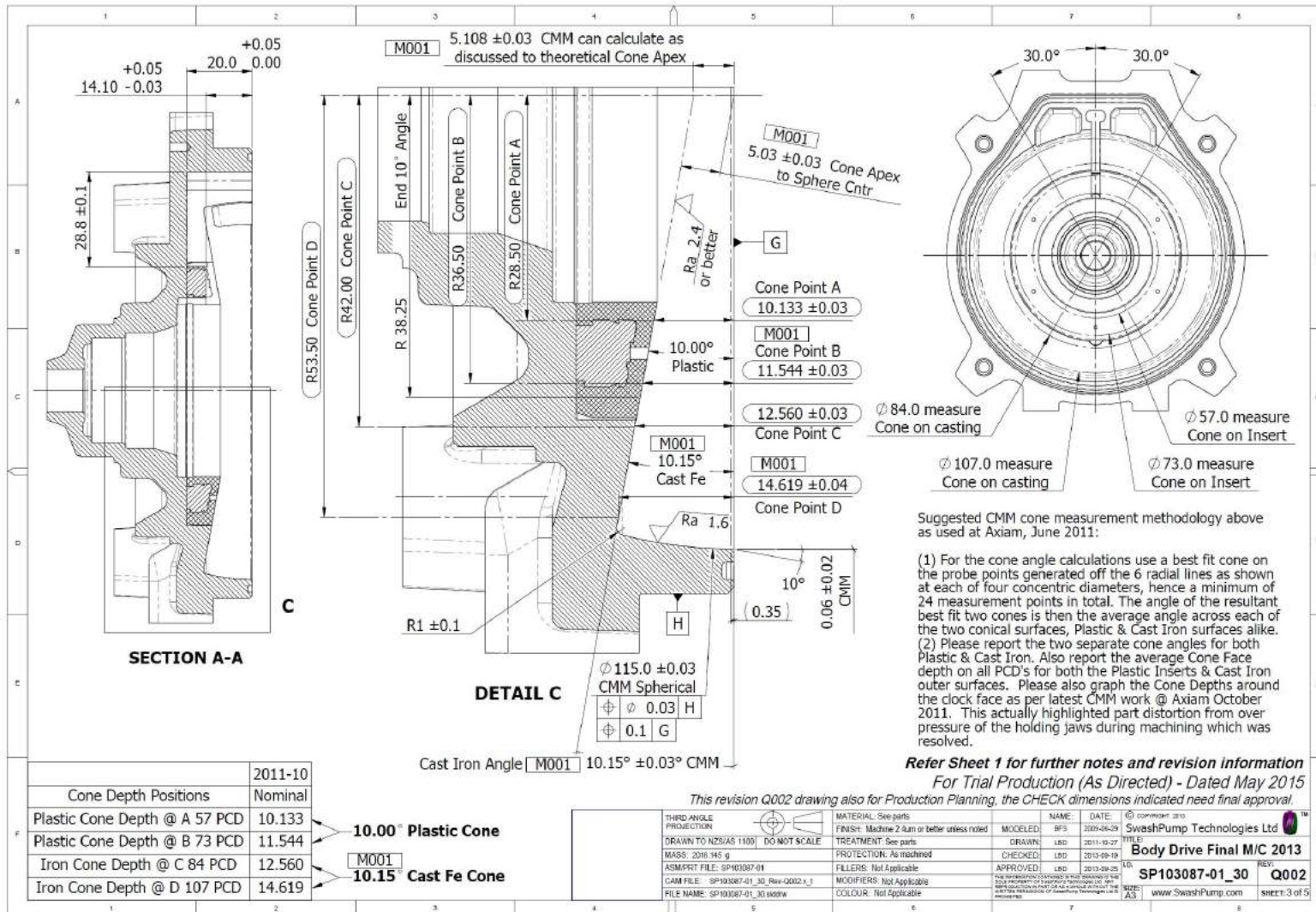


Drawings & QA



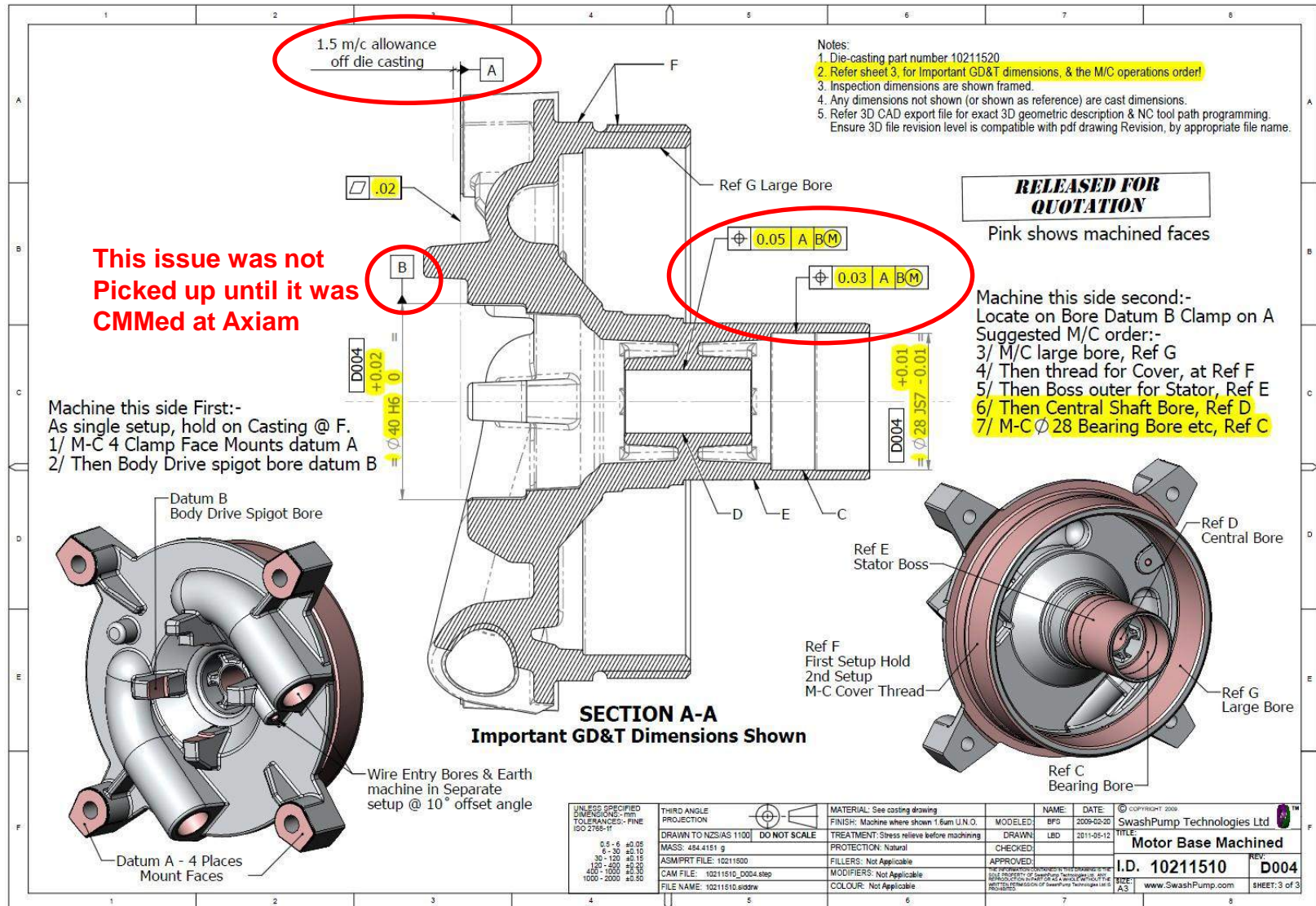
Released on 2013-09-25 4:30:00 p.m. Released for Production Planning !

Drawings & QA



Released on 2013-09-25 4:30:00 p.m. Released for Production Planning!

WDT Motor Base Bearing Concentricity Problem



CMM of WDT Motor Base Bearing



Drive Side Body / Motor Alignment check

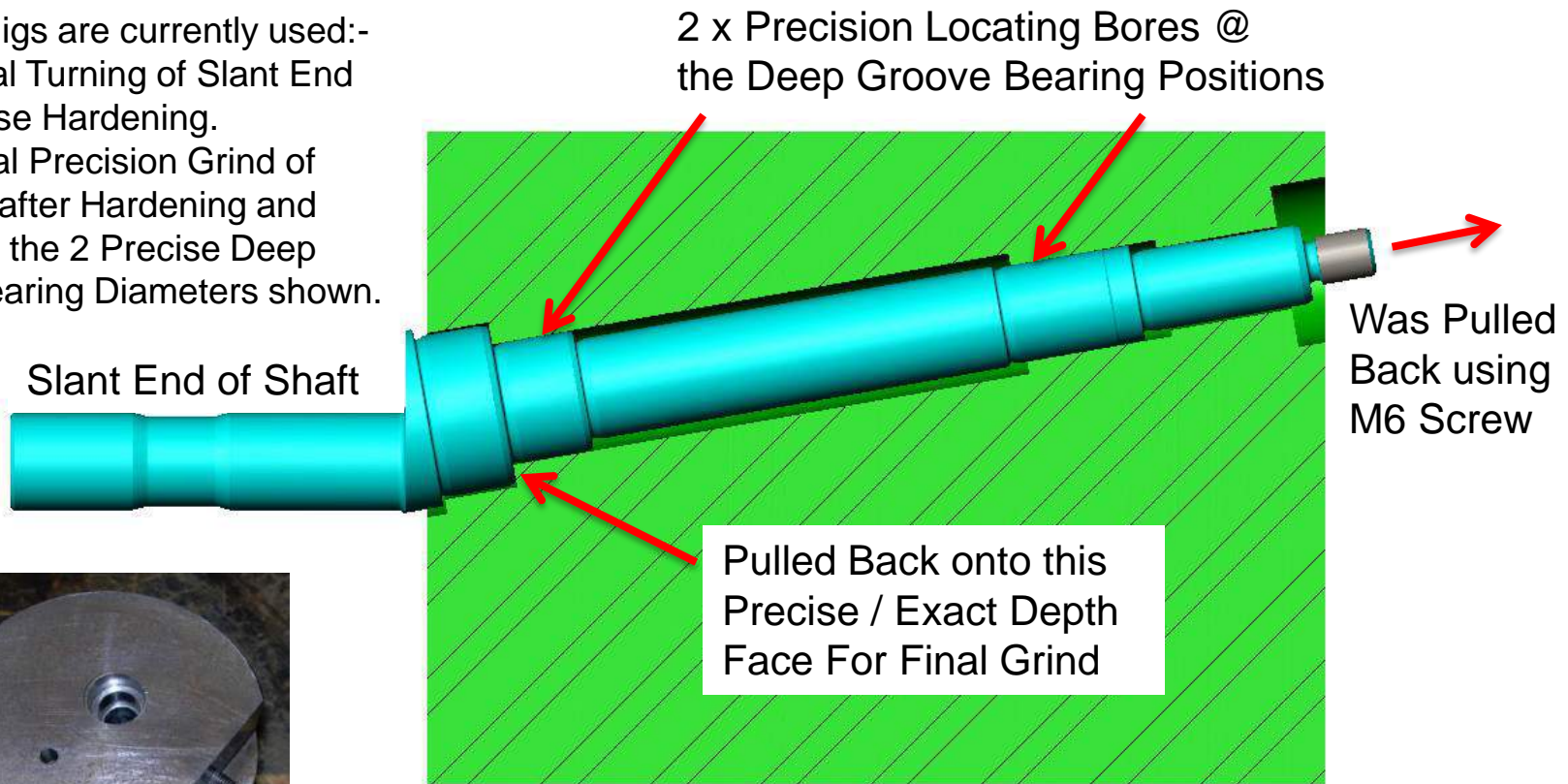
We made an Alignment verification Jig at Axiam, so we could double check without CMM



The Manual Drive Shaft Jig(s) in Cross Section

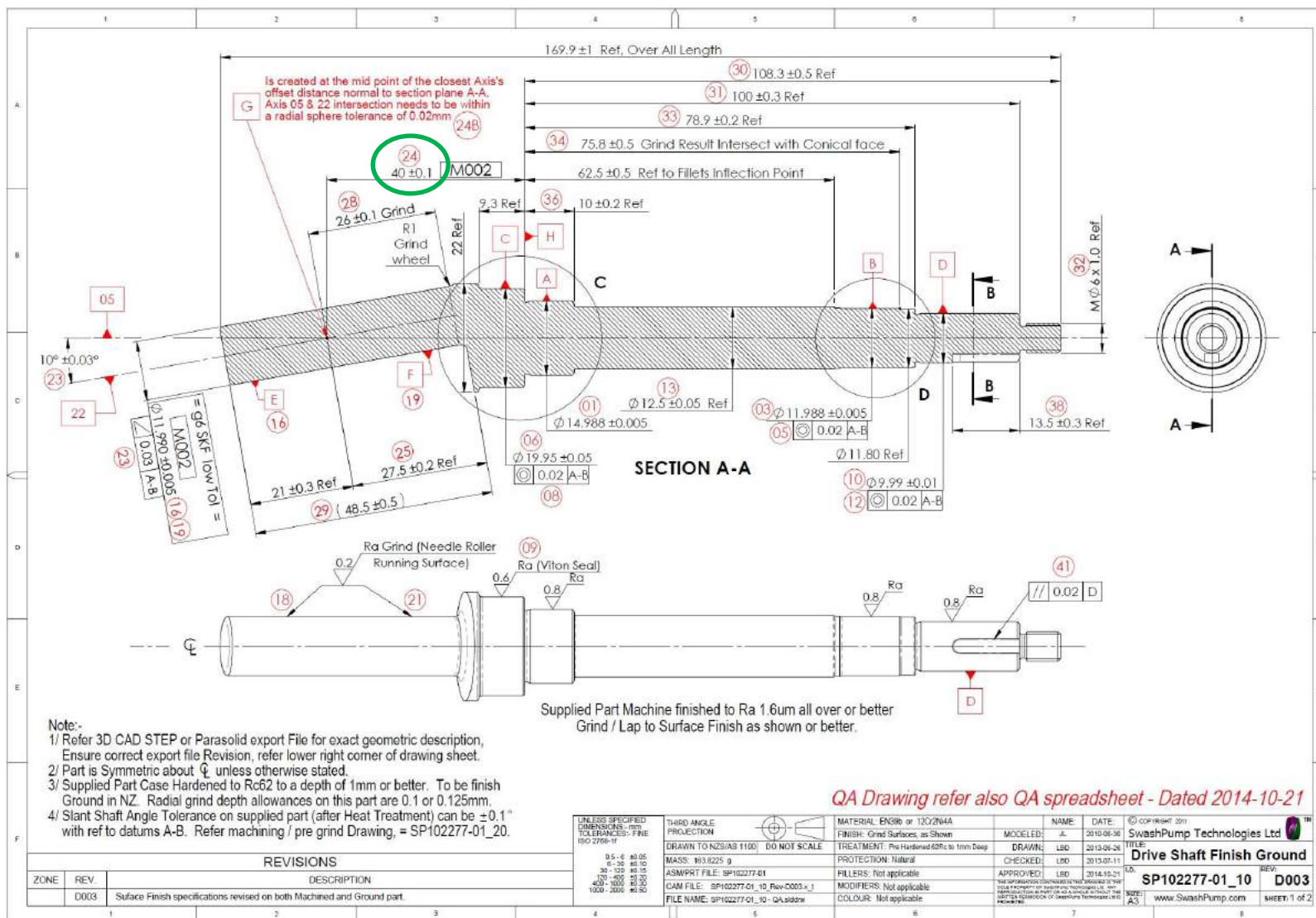
Full Production manufacture would be on an NC grinder with hydraulic self centring collets

Note two Jigs are currently used:-
1st for Initial Turning of Slant End Before Case Hardening.
2nd for Final Precision Grind of Slant End after Hardening and grinding of the 2 Precise Deep Groove Bearing Diameters shown.

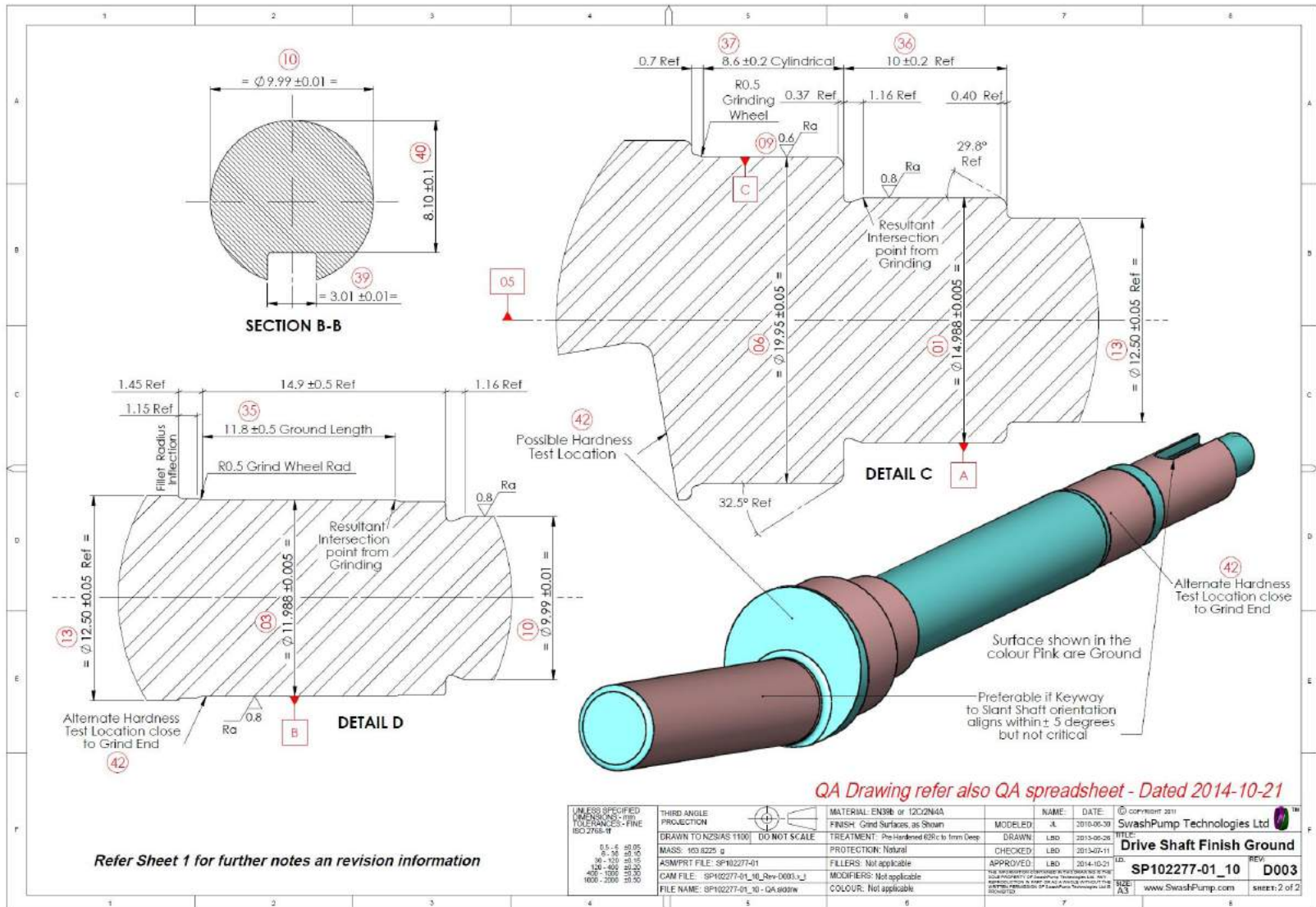


SECTION A-A

Shaft also needed QA with CMM



Shaft also needed QA with CMM



Detailed Measurement CMM Report, Body Drive

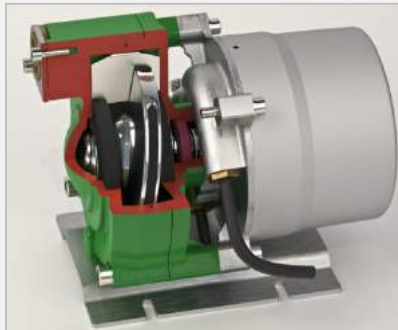
Name:		Batch:	ACCEPTED / REJECTED / CONCESSSED (delete as applicable)
Date:			

DATA																																																												
		Predicated						Std																																																				
		LSL	MD	USL	LL	UL	Cpk	Pp	Nmean	Dov	Min	Max																																																
1	Total body length	62.300	62.500	62.700	62.423	62.991	1.336	1.336	62.507	0.408	62.458	62.535																																																
2	M3 Thread length	10.000	11.000	12.000	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	0.000	0.000																																																
3	M3 Thread length	16.000	16.500	17.000	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	0.000	0.000																																																
4	M3 thread to OH																																																											
5	Divider slot side to centre	2.015	2.025	2.035	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	0.000	0.000																																																
6	Divider slot width	4.030	4.050	4.070	4.029	4.076	0.735	0.836	4.053	0.008	4.030	4.064																																																
7	Divider slot depth to G				0.017	0.017		0.822	0.005	0.004	0.000	0.015																																																
8	Divider slot height to G	20.900	20.925	20.950	19.972	20.367	0.341	0.585	20.915	0.014	19.985	20.032																																																
9	Divider slot height	66.300	66.400	66.500	66.278	66.427	0.707	0.336	66.352	0.025	66.302	66.385																																																
10	G to D distance	41.800	42.300	42.800	41.878	42.053	0.751	1.141	41.968	0.029	41.911	42.002																																																
11	O to D parallel			0.030	0.119	0.119	0.201	0.045	0.025	0.000	0.078																																																	
12	Main spigot dia E	39.960	39.960	40.000	39.979	40.004	0.691	0.807	39.991	0.004	39.980	39.996																																																
13	main spigot dia H	126.270	126.285	126.300	126.248	126.314	0.312	0.488	126.280	0.011	126.255	126.295																																																
14	H conc to E			0.020	0.044	0.044	0.552	0.026	0.000	0.042	0.035																																																	
15	Bearing Bore dia J	32.000	32.008	32.016	31.981	32.020	0.037	0.414	32.001	0.000	31.990	32.013																																																
16	J conc to E			0.020	0.026	0.026	0.714	0.012	0.005	0.005	0.021																																																	
17	Viton seal bore dia	35.000	35.020	35.040	34.982	35.019	0.843	1.036	35.001	0.000	34.981	35.012																																																
18	Viton seal bore conc to J			0.030	0.012	0.012	0.804	0.004	0.003	0.001	0.011																																																	
19	Spherical seal bore diameter	52.550	52.570	52.590	52.540	52.601	0.649	0.655	52.570	0.010	52.530	52.587																																																
20	Spherical seal conc to J			0.030	0.035	0.035	0.703	0.016	0.000	0.000	0.029																																																	
21	Shaft hole bore diameter	12.750	12.800	12.850	12.753	12.860	0.738	0.706	12.790	0.021	12.763	12.823																																																
22	Shaft hole conc to J			0.020	0.013	0.013	0.729	0.005	0.003	0.001	0.010																																																	
23	Spigot step to face plane G	11.900	12.000	12.100	11.911	12.036	1.132	1.059	11.974	0.021	11.921	11.991																																																
24	Bearing shoulder to G	48.950	49.000	49.050	48.933	49.048	0.701	0.800	48.990	0.019	48.950	49.015																																																
25	Shaft hole entry face to O	50.800	51.000	51.200	50.805	51.211	0.940	0.905	50.908	0.008	50.831	51.177																																																
26	Cone depth at 94mm diameter	12.524	12.544	12.564	12.489	12.589	0.293	0.408	12.530	0.016	12.504	12.566																																																
27	Cone angle at 24mm points	119.970	160.000	160.000	159.973	160.242	-0.147	0.153	160.000	0.002	159.980	160.129																																																
28	Main sphere diameter	114.960	115.000	115.020	114.964	115.016	0.438	0.600	114.981	0.008	114.964	115.001																																																
29	Main sphere conc to H			0.030	0.024	0.024	1.966	0.000	0.003	0.003	0.011																																																	
30	Main sphere centre to G			0.100	0.116	0.116	0.576	0.029	0.000	0.000	0.110																																																	
31	Sphere edge chamfer	0.040	0.060	0.080	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	0.000	0.000																																																
32a	Left top face height to G	73.630	73.650	73.670	73.619	73.683	0.416	0.800	73.641	0.007	73.623	73.653																																																
32b	Right top face height to G	73.630	73.650	73.670	73.626	73.681	0.779	1.336	73.644	0.006	73.631	73.653																																																
33	Left face at 50mm	37.640	37.650	37.660	37.563	37.687	0.416	0.600	37.633	0.011	37.634	37.671																																																
34	Left face angle	123.470	123.500	123.530	123.453	123.548	0.318	1.954	123.489	0.009	123.466	123.499																																																
35	Right face at 50mm	37.640	37.650	37.660	37.563	37.687	0.350	0.752	37.631	0.009	37.612	37.645																																																
36	Right face angle	56.470	56.500	56.530	56.489	56.539	0.656	1.096	56.500	0.000	56.490	56.527																																																
37	Foot plane height to centre	70.800	70.700	70.700	70.740	70.740	-0.233	0.311	70.725	0.008	70.714	70.745																																																
38	Foot plane angle to K	89.970	90.000	90.030	89.999	90.003	12.574	12.583	90.000	0.001	89.990	90.002																																																
39	Plane D perp to E			0.022	0.022	0.022	0.848	0.010	0.004	0.001	0.014																																																	
40	Wave spring face to G	22.850	23.000	23.150	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	0.000	0.000																																																
41	Top left spigot radius	9.950	9.900	9.950	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	0.000	0.000																																																
42	Top right spigot radius	9.950	9.900	9.950	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	#DIV/0!	0.000	0.000																																																
43	Top spigot face perp to K			0.040	0.014	0.014	2.176	0.005	0.003	0.000	0.011																																																	
44	Cone angle to G	5.118	5.148	5.178	5.076	5.210	0.402	0.456	5.144	0.022	5.105	5.189																																																
45	Cone conicity			0.078	0.056	0.056	0.000	0.006	0.007	0.009	0.049																																																	

TUV Approvals

- First Pump to be certified to new EU legislation
- First **two sided pump** to be certified with Petrol instead of Air
- Certifications
 - Zephyr 2 Pump with 2x (burket control valves, OPW nozzles and PEC hoses)
 - Zephyr 2 Pump with 2x (burket control valves, Elaflex nozzles and PEC hoses)

Swash Vapor Recovery System (VRS)



Prototype



Production Unit



IECEx

SwashPump (Pre) Production Products

Gas

Swash Vapor Recovery System (VRS)



Prototype



Production Unit



System testing
SPT lab Auckland

Liquid

Swash Fuel Pump System (FPS) Petrol, Diesel, Bio Fuel



Prototype



Production Unit



In Dispenser testing
Auckland

SwashVRS - Petrol Vapour Recovery System



The End

TMI - beyond here

Design SMART[®] *Ltd*

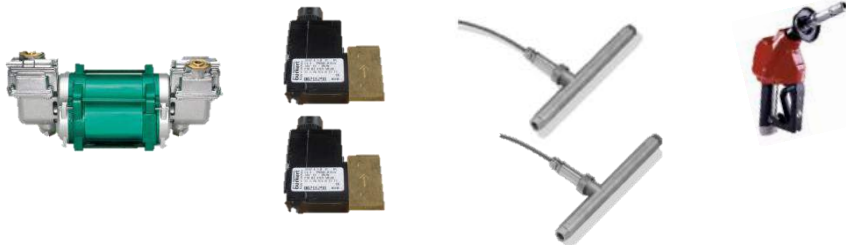
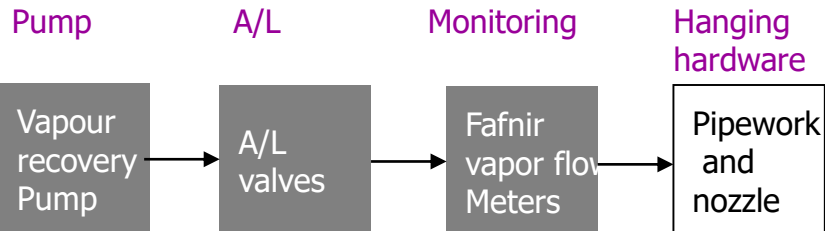
Innovative Mechanical Engineering

www.DesignSMART.co.nz

Summary

- ➔ Some countries vapor recovery legislation requires the vapour flow to be measured and compared to the volume of liquid dispensed to calculate the A/L ratio for each fill cycle.

Current monitored system



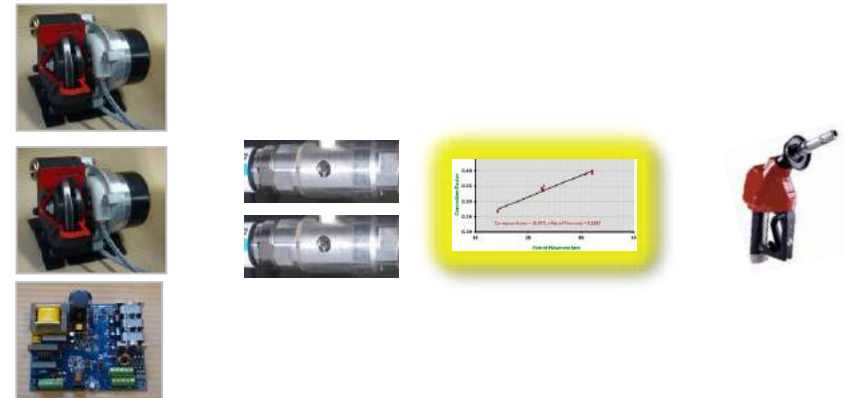
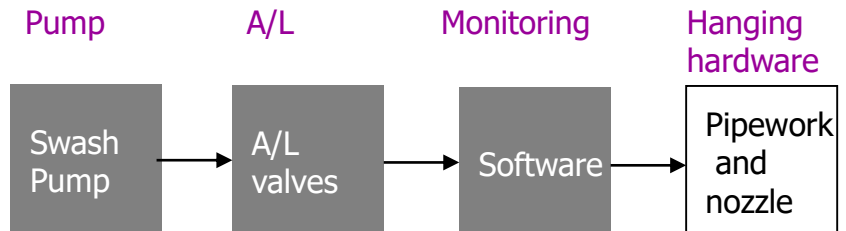
\$500

\$1500

\$3000

Cost for 2 fill points : \$5000 (not including hanging hardware)

SPT / OPW monitored system



\$700

\$200

Free –
Included in
controller

Cost for 2 fill points : \$900 (not including hanging hardware)

Superior Performance v Roots

Definition	SwashPump™
Dry Pump	Yes
RPM	Low i.e. 980 to 1440 for 4 or 6 pole synchronous motor speed
Scaleable	200-50,000 NI/min 7-1750 ft ³ /min @ 500mbar
Thermo-dynamic Efficiency	>55% @ 500 mbar Adiabatic Reversible
Exhaust Temperature	< 92 C <194 F
Noise Level	< 85 dB(A)
Vibration	< 5.6 mm/s RMS <.02 ft/s RMS
Service Interval	Trunnion Maintenance approx 2000 hours

Test Rig & Dynamometer



Joel on PV for Roots Blower & Theory

➡ Joel, Roots Blower PV

398 AIR COMPRESSORS

driven together through external gearing. Rotors with more than two lobes are sometimes used when an increase in pressure ratio is required. The rotors rotate in a casing. The operation is as follows. Gas is taken in at the intake and as the rotors are rotated so a mass of gas of volume V becomes trapped between the rotor and the casing. This volume V is

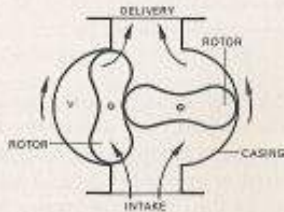


Fig. 122

transported to the delivery side of the machine. As the mass of gas is delivered so compressed gas already on the delivery side compresses the mass to delivery pressure. The volume of gas transported is $4V$ per revolution. The pressure ratio through the machine is usually low, say 2:1.

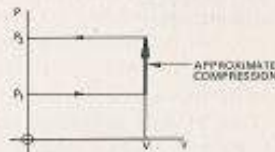


Fig. 123

An approximate P - V diagram for the Roots blower is given in Fig. 123.

➡ Joel, Theoretical Compression

ROTARY AIR COMPRESSORS 397

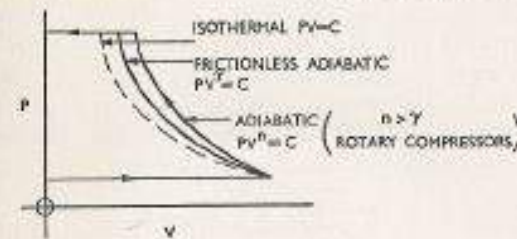


Fig. 121

Ideal SwashPump PV diagram, Closer to the Ideal Isentropic Process

Swashpump P2.5 ideal pv Diagram with non return valves on the outlet

Swashpump Technologies Page 1 of 2

Author: Ben Smit

6/08/2007

Version: 1.0

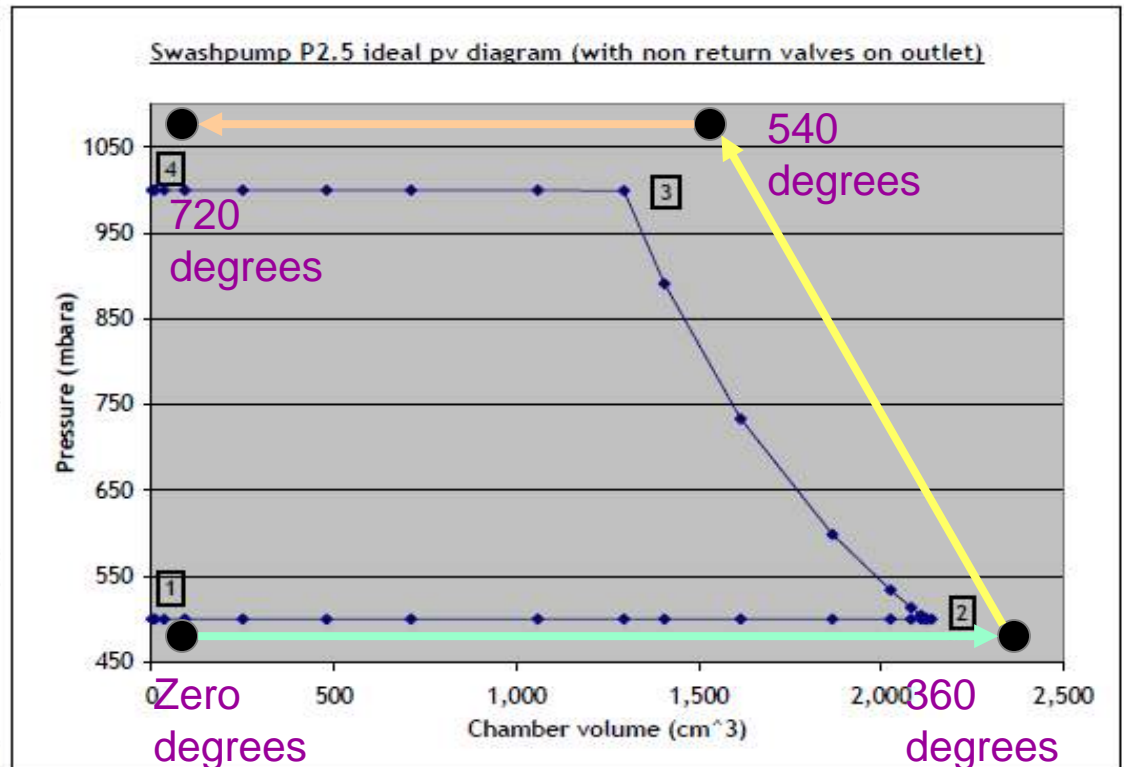
Shaft angle (Degrees)	Chamber volume (cm ³)	Pressure (mbara)	
0	0	500	1
29	9	500	
50	34	500	
70	91	500	
100	250	500	
130	480	500	
150	711	500	
180	1,059	500	
	1,295	500	
210	1,406	500	
240	1,615	500	2
260	1,867	500	
290	2,026	500	
310	2,083	500	
329	2,110	500	
340	2,124	500	
360	2,138	500	
380	2,124	500	
389	2,110	504.7	
410	2,083	513.8	3
430	2,026	534.2	
460	1,867	598.9	
490	1,615	733.7	
510	1,406	890.9	
	1,295	999.6	
540	1,059	1000	
570	711	1000	
600	480	1000	
620	250	1000	
650	91	1000	4
670	34	1000	
689	9	1000	
720	0	1000	

Intake cycle - inlet port open

Adiabatic compression $PV^{1.4}$

Expulsion past the reed valves

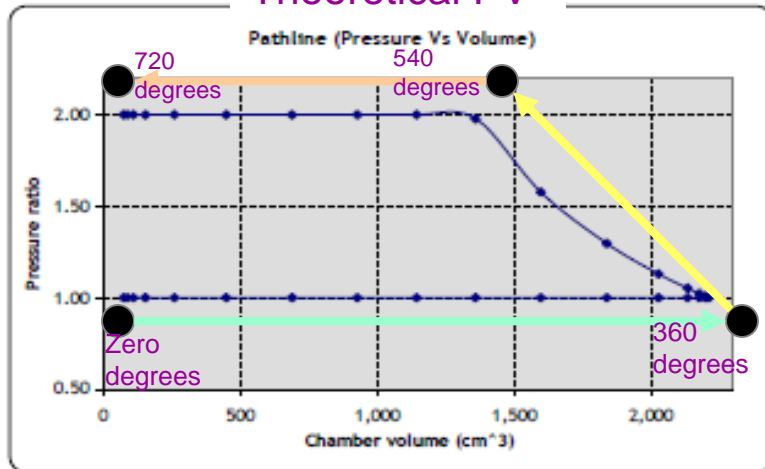
Full pumping cycle



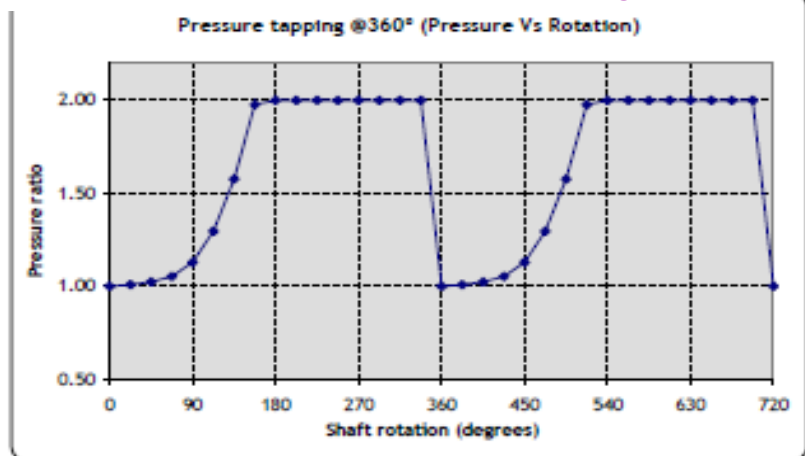
Note: There are two chambers in one swashpump.

Theoretical SwashPump PV diagram vs. Actual SwashPump PV diagram

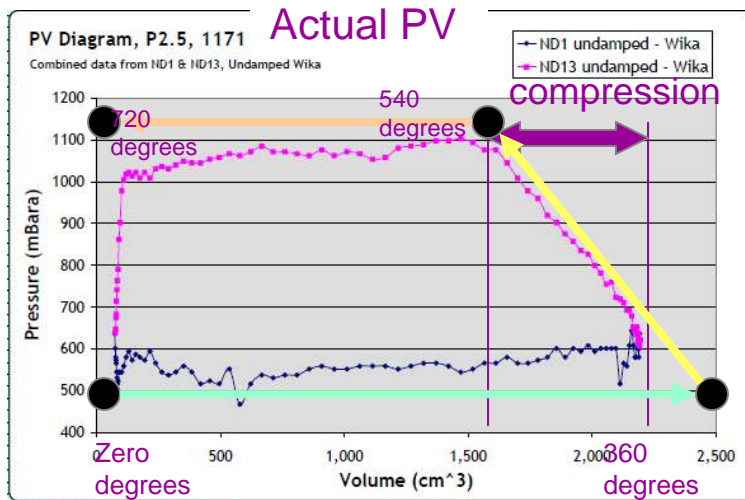
Theoretical PV



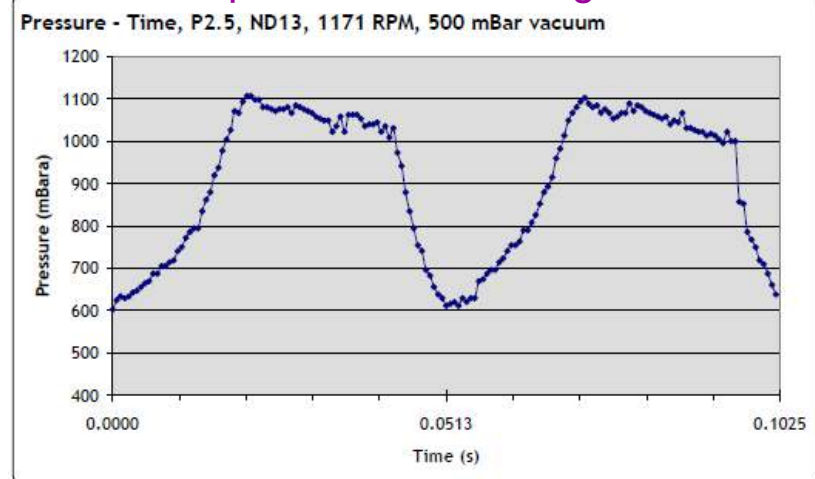
Two expulsions occur in 720 degrees



Actual PV



Actual expulsions in 720 degrees



Materials Considerations

Comparison of possible construction materials

Material	Description	Density Kg/m ³	CTE x10E-6 m/m.K	k W/m.K	E Gpa	Fy Mpa	Ni %	Cr %	Cost / Kg \$	Caustic	Castability	Machinability	Heta treatment & Grinding Req'd?
Ni Resist	As previously prototyped	7400	18.7	13	150	205	20%	3%	\$ 6.60	B	C	C	No
StStl AISI 420	Martinsitc Stainless steel	7800	10.3	25	200	345	0%	13%		C	C	C	Yes
StStl AISI 2205	Duplex Stainless steel	7800	13.0	14	200	450	8%	23%		A	D	D	No
High Chromium Iron	Wear resistant grade of cast iron	7700	10.0				1%	12%		D	D	C	Yes
Aluminium LM13	Cast aluminium	2700	19.0	167	73	160	2%	N/A	\$ 6.00	F	B	A	No
Ni Resist Type D-5		7700	5.0			200	35%	0%	\$ 10.00				
Grey cast iron		7300	11.0	52	150	200	0%	1%	\$ 2.00				No
PPS	High Temp Plastic	1.5	20.0		15	120	0%	0%	\$ 30.00	B	A	A	No
Titanium-6AL-4Va	High tech!	4500	9.2	6	117	760	0%	0%	\$ 71.00	A			No

Density	Density	How heavy it is.
CTE	Coefficient of thermal expansion at 20 ° C	How much it expands with temperature.
k	Coefficient of thermal conductivity	How quickly it conducts heat.
E	Youngs modulus	How rigid it is.
Fy	Yield strength	How strong it is.
% Ni	Percentage of Nickel in the material	Big determinant of material cost
% Cr	Percentage of Chromium in the material	Big determinant of corrosion resistance
Caustic	Corrosion resistance to caustic cleaning solution	Graded from A (excellent) to F (Poor)
Castability	Castability	Graded from A (excellent) to F (Poor)
Machinability	Machinability	Graded from A (excellent) to F (Poor)
Heta treatment & ...	Heat treatment req'd to develop full strength of the material. May not be a requirement, as stiffness is important, not strength.	